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# TRAJECTORY GENERATION FOR THE N-TRAILER PROBLEM USING GOURSAT NORMAL FORM 

by<br>D. Tilbury, R. Murray, and S. Sastry

Memorandum No. UCB/ERL M93/12
15 February 1993

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# Trajectory Generation for the $N$-trailer problem using Goursat Normal Form* 

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#### Abstract

In this paper, we develop the machinery of exterior differential forms, more particularly the Goursat normal form for a Pfaffian system, for solving nonholonomic motion planning problems, i.e. planning problems with non-integrable velocity constraints. We apply this technique to solving the problem of steering a mobile robot with $n$ trailers. We present an algorithm for finding a family of transformations which will display the given system of rolling constraints on the wheels of the robot with $n$ trailers in the Goursat canonical form. Two of these transformations are studied in detail. The Goursat normal form for exterior differential systems is dual to the so-called chained form for vector fields tbat we have studied in our earlier work. Consequently, we are able to give the state feedback law and change of coordinates to convert the N -trailer system into chained form. Three methods for steering chained form systems using sinusoids, piecewise constants and polynomials as inputs are presented.

The motion planning strategy is therefore to first convert the $N$-trailer system into chained form, steer the corresponding chained form system, then transform the resulting trajectory back into the original coordinates. Simulations and frames of movie animations of the $N$-trailer system for parallel parking and backing into a loading dock using this strategy are also included.


[^0]
## 1. Introduction

In the past few years there has been a great deal of interest in the generation of motion planning algorithms for robots with nonholonomic or non-integrable velocity constraints in cluttered environments. These constraints on the instantaneous velocities that can be achieved arise from the kinematics of the drive mechanisms of the carts. This work has been a departure from the traditional robot motion planning (see for example [6, 15, 18]) which concentrated on understanding the complexity of the computational effort associated with planning trajectories for robots (with no constraints on their instantaneous velocities) which would avoid both fixed and moving obstacles. Unfortunately the motion plans arising from these more traditional methods often required sideways motion of robot carts with wheels, and as was pointed out by Laumond, most mobile robots are not on castors [19, 20].

In this paper, we consider and solve the motion planning problem for a system consisting of a car-like mobile robot pulling $n$ trailers. This system has been an important canonical example for the work on nonholonomic motion planning ever since it was posed in [22, 28]. The nonholonomic constraints for this system arise from constraining each pair of wheels to roll without slipping. Strictly speaking, if an axle has a differential that keeps the pair of wheels rolling without slipping, then each wheel turns a different amount in accordance with a simple geometric relationship called the Alexander-Maddocks condition [1]. In our system we will neglect this and model the wheels on an axle as being parallel.

The system of a car with $n$ trailers has been viewed as a canonical example because each trailer adds one dimension to the state space of the system (representing its angle with respect to the inertial frame) and one nonholonomic constraint. Regardless of the number of trailers attached, the general system always has two degrees of freedom, corresponding to the driving and steering directions of the front car. It has been shown that every point in the state space is reachable, i.e. that the system is completely controllable [22]. The question that is answered in this paper is one of constructive controllability; explicit open loop controls for steering the car with $n$ trailers from an initial to a final position are given.

We first give a brief description of some of the previous work on this problem. A more detailed review of the general nonholonomic motion planning problem can be found in [29] or in a recent collection of papers [24]. A more detailed description of the $N$-trailer problem and its variations can be found in [21].

Barraquand and Latombe [2] proposed a planner for cars with trailers in a cluttered environment, with an attempt at finding one with a minimal number of backups. The main drawback to their approach was that it required a discretization of the state space followed by an exhaustive search of all possible directions the robot could go at each point. Consequently, the method became computationally infeasible for a large number of trailers. The method, however, worked well in a very cluttered environment since the presence of many obstacles drastically reduced the number of search directions. Related to this work is that of Divelbiss and Wen [8] which uses gradient descent in a discretized input space. Obstacles can be incorpo-
rated into their method using potential fields. The convergence properties of their methods are currently under investigation.

Reeds and Shepp [30] proved an interesting result on the minimum length feasible paths for a robot of the Hilare type with bounded turning radius. They showed that the optimal length path belonged to a family of paths that consisted of segments of straight lines and arcs of circles. It seems doubtful that such a method could be generalized easily to a car with $n$ trailers.

A paper by Murray and Sastry [29] studied motion planning for nonholonomic systems, and focused attention on a specific class of systems in so-called "chained form":

$$
\begin{aligned}
\dot{x}_{1} & =u_{1} \\
\dot{x}_{2} & =u_{2} \\
\dot{x}_{3} & =x_{2} u_{1} \\
& \vdots \\
\dot{x}_{n} & =x_{n-1} u_{1} .
\end{aligned}
$$

This class of systems was inspired by some early work of Brockett [3] on optimal control of "canonical systems". In [29], we gave sufficient conditions for converting systems into chained form, and an algorithm (using sinusoids at integrally related frequencies) for steering chained form systems. The theory was used to transform the front-wheel drive car, a car with one trailer, and a hopping robot into chained form, and to find feasible trajectories for these systems using the sinusoidal steering algorithm. However, the car with two trailers did not fit the sufficient conditions and was left an open problem. Recently Sørdalen [32] showed that the system of the car with $n$ trailers could be put into chained form using the coordinates of the $n^{\text {th }}$ trailer (rather than those of the cab) for parameterizing the configuration space of the system.

Fernandes, Gurvits and Li [9] used numerical methods for solving constrained optimal control problems associated with nonholonomic motion planning problems, using a perturbation (of the cost functional) to make the singular optimal control problem regular. In other work [10], they also suggested the use of input sinusoids (as basis functions) in a Ritz approximation algorithm for steering nonholonomic systems.

Sinusoids were also used in a method proposed by Sussmann and Liu [33], see also Gurvits and Li [14]. Their method was completely general in that it applied to any controllable nonholonomic system, and used asymptotically high frequency, high magnitude sinusoids to achieve convergence. This method was applied to the system of Hilare with two trailers in [35], but the paths generated were highly oscillatory and impractical, owing to their use of high magnitude and frequency sinusoids.

Divelbiss and Wen [8] have explored a computational approach to the N -trailer problem, in which they discretize the system and use gradient descent in the (discretized) input space to generate a feasible path. Convergence properties of their
method are currently under investigation, but the algorithm has demonstrated good performance in simulation. They are also able to incorporate obstables into the problem formulation, using potential field methods.

Several other methods have been proposed which used piecewise constant inputs
$\Longrightarrow$ a
 [16, 17, 26]. All of these worked best in the cases where the control Lie algebra is nilpotent. Their extension to systems whose control Lie algebra is not nilpotent is not fully satisfactory, requiring a large number of steps to come close to the goal point.

Our current paper is some what different in style from most of the previous work. Instead of focusing on the directions in which the system is allowed to move, namely the two vector fields which correspond to the two degrees of freedom of the system, we have defined the system from the constraints on its velocity. That is, instead of looking at the control system

$$
\dot{x}=g_{1}(x) u_{1}+g_{2}(x) u_{2}
$$

and the distribution spanned by the input vector fields $\Delta=\left\{g_{1}, g_{2}\right\}$, we consider the exterior differential system orthogonal to this distribution, namely $I=\Delta^{\perp}=$ $\left\{\alpha^{1}, \ldots, \alpha^{n-2}\right\}$. In the context of motion planning, this is in some sense a very natural framework since each $\alpha^{i}$ is a one-form defined on the tangent space to the configuration space, and represents the constraint that the wheels on the $i^{\text {th }}$ axle must roll without slipping.

This system $I$ is called a Pfaffian system (of codimension 2); such exterior differential systems and their properties were first studied by Pfaff in the early 1800's. There exists a large body of work on Pfaff's problem in the literature (see [4] for a historical overview). The formulation of the $N$-trailer problem as an exterior differential system allows us to draw on classical results by Goursat, Engel, Cartan, and others on classification and canonical forms. Most of the relevant results in this area are presented in abbreviated fashion in [4] and are reviewed in Section 2 of this paper. The normal form for Pfaffian systems of codimension 2 that was proposed by Goursat is in fact the dual of chained canonical form as defined above. As in the work of Sørdalen, the calculations for the Goursat normal form are simplified quite considerably by using the coordinates of the last trailer instead of those of the cab to parameterize the configuration space of the multi-trailer system.

After the crash course on exterior differential systems in Section 2, we examine the Pfaffian system associated with a mobile robot towing $n$ trailers. We show in Section 3 that this system can be converted into Goursat's normal form or equivalently chained form. Section 4 is devoted to presenting methods for steering systems in chained form. Three different methods are presented using as inputs sinusoids, piecewise constant inputs (as in [26]) and polynomials. Finally, we apply some of these steering methods to the $N$-trailers example, and display the results in Section 5. There are movie animations of two of the trajectories; a two-trailer system can be seen parallel-parking by viewing the upper right-hand corner of the odd numbered pages (from the front of the paper to the back), and this same system backing into a loading dock can be seen in the upper left-hand corner of the even

numbered pages (from the back of the paper to the front).

## 2. A Crash Course on Exterior Differential Systems

In this section we give a review of the tools available from the study of exterior differential systems and show how to apply these tools to the problem of finding a feedback transformation which converts a system into chained form. We present only a very brief review of the necessary tools here, concentrating on the computations that must be performed. A much more detailed description can be found in the monograph by Bryant et al. [4].
2.1. Exterior algebra. Let $V$ be a vector space over $\mathbb{R}$, which we also refer to using the notation $\Lambda^{1}$. We define a new vector space $\Lambda^{2}$ by defining the wedge product as a skew-symmetric bilinear map which satisfies:

$$
\begin{align*}
\left(a_{1} \alpha_{1}+a_{2} \alpha_{2}\right) \wedge \beta & =a_{1}\left(\alpha_{1} \wedge \beta\right)+a_{2}\left(\alpha_{2} \wedge \beta\right) \\
\alpha \wedge\left(b_{1} \beta_{1}+b_{2} \beta_{2}\right) & =b_{1}\left(\alpha \wedge \beta_{1}\right)+b_{2}\left(\alpha \wedge \beta_{2}\right) \\
\alpha \wedge \alpha & =0  \tag{1}\\
\alpha \wedge \beta & =-\beta \wedge \alpha .
\end{align*}
$$

That is, $\wedge$ is a bilinear, associative, distributive, non-commutative product mapping $\Lambda^{1} \times \Lambda^{1} \rightarrow \Lambda^{2}$. If $\left\{\sigma_{i}\right\}$ is a basis for $\Lambda^{1}$, then $\sigma_{i} \wedge \sigma_{j}, 1 \leq i<j \leq n$ is a basis for $\Lambda^{2}$. It follows that the dimension of of $\Lambda^{2}$ is ( $\left.\begin{array}{c}n \\ 2\end{array}\right)$. An element of $\Lambda^{2}$ is called a two-vector.

In a similar way, we define a $p$-vector $\theta \in \Lambda^{p}$ by taking the wedge product between $p$ one-vectors and using the rules

$$
\begin{gathered}
(a \alpha+b \beta) \wedge \alpha_{2} \wedge \cdots \wedge \alpha_{p}=a \alpha \wedge \alpha_{2} \wedge \cdots \wedge \alpha_{p}+b \beta \wedge \alpha_{2} \wedge \cdots \wedge \alpha_{p} \\
\alpha_{1} \wedge \cdots \wedge \alpha_{p}=0 \text { if any } \alpha_{i}=\alpha_{j}, i \neq j \\
\alpha_{1} \wedge \cdots \wedge \alpha_{p} \text { changes sign if any two } \alpha_{i} \text { are interchanged. }
\end{gathered}
$$

$\Lambda^{p}$ consists of all $p^{\text {th }}$ order exterior products and has a basis given by $\left\{\sigma_{h_{1}} \wedge \cdots \wedge \sigma_{h_{p}}\right\}$ where $\left\{\sigma_{i}\right\}$ is a basis for $\Lambda^{1}$ and the $h_{i}$ 's are ordered. $\Lambda^{p}$ is a vector space of dimension $\binom{n}{p}$. In particular, $\operatorname{dim} \Lambda^{n}=1$ and $\operatorname{dim} \Lambda^{k}=0$ if $k>n$. For completeness, we define the set of zero-vectors as $\Lambda^{0}=\mathbb{R}$.

The wedge product is a very powerful tool which can be used to great advantage in calculations. We will make frequent use of the following facts:

Proposition 1. The vectors $v_{1}, \ldots, v_{p} \in \Lambda^{1}$ are linearly dependent if and only if $v_{1} \wedge \cdots \wedge v_{p}=0$.

Corollary 1.1. Let $\xi, v_{i} \in \Lambda^{1}$. If $\xi \wedge x_{1} \wedge \cdots \wedge v_{p}=0$ then

$$
\xi=\sum_{i=1}^{p} \alpha_{i} v_{i} \quad \alpha_{i} \in \mathbb{R}
$$

Just as in the case of polynomials, it is often desirable to speak of a vector of mixed order (or unknown order). Using the wedge product, one can define an algebra over the set of exterior forms. Let $\Lambda=\Lambda^{0} \oplus \Lambda^{1} \oplus \cdots \oplus \Lambda^{n}$ and define multiplication between two elements of $\Lambda$ using the wedge product. The wedge product is a
 bilinear, associative, distributive, skew product which maps $\Lambda^{r} \times \Lambda^{s} \rightarrow \Lambda^{r+s}$ and hence $\Lambda \times \Lambda \rightarrow \Lambda$. We say an element $\xi \in \Lambda$ is homogeneous of order $p$ if $\xi \in \Lambda^{p}$; i.e. it is a $p$-vector. A exterior form is non-homogeneous if it has components of different orders.

If $V$ is a vector space of dimension $n$, its dual, $V^{*}$, is also a vector space of dimension $n$. The exterior product over $V^{*}$ can be used to form the vector space $\Omega^{p}(V):=\Lambda^{p}\left(V^{*}\right)$. An element $\alpha \in \Omega^{p}$ is called a $p$-form.
2.2. Differential forms. Given a manifold $M$ of dimension $n$, the tangent space of $M$ at a point $x$ is a vector space of dimension $n$, denoted $T_{x} M$. The vector space $\Lambda^{p}\left(T_{x} M\right)$ consists of all $p$-vectors constructed from tangent vectors in $T_{x} M$. By attaching the vector space $\Lambda^{p}\left(T_{x} M\right)$ to each point $x \in M$, we get a bundle structure on $M$, which we write as $\Lambda^{p}(M)$. Similarly, the bundle $\Omega^{p}(M)$ is defined by using the dual space $T_{x}^{*} M$. We call a element $\omega \in \Omega^{p}(M)$ an exterior differential $p$-form on $M$.

Relative to a local coordinate chart, we describe the tangent and cotangent bundles by choosing a local basis:

$$
\begin{aligned}
& T_{x} M=\operatorname{span}\left\{\frac{\partial}{\partial x_{1}}, \ldots, \frac{\partial}{\partial x_{n}}\right\} \\
& T_{x}^{*} M=\operatorname{span}\left\{d x_{1}, \ldots, d x_{n}\right\}
\end{aligned}
$$

where

$$
d x_{i}\left(\frac{\partial}{\partial x_{j}}\right)=\delta_{i j} .
$$

A $p$-form $\omega$ on $M$ can be represented in this basis as

$$
\omega(x)=\sum_{i_{1}<\cdots<i_{p}} \omega_{i_{1} \cdots i_{p}}(x) d x_{i_{1}} \wedge \cdots \wedge d x_{i_{p}} .
$$

We say that $\omega$ is smooth if the coefficient functions $\omega_{i_{1} \ldots i_{p}}$ are smooth functions of $x$ for any choice of coordinate chart.

Let $\Omega(M)$ be the algebra of exterior differential forms on $M$. The exterior derivative on $\Omega(M)$ is the unique map $d: \Omega^{r} \rightarrow \Omega^{r+1}$ which satisfies the following properties:
(1) If $f \in \Omega^{0}(M)=C^{\infty}(M)$ then $d f=\sum \frac{\partial f}{\partial x_{i}} d x_{i}$ (relative to a local coordinate chart).
(2) If $\theta \in \Omega^{r}, \sigma \in \Omega^{s}$ then $d(\theta \wedge \sigma)=d \theta \wedge \sigma+(-1)^{r} \theta \wedge d \sigma$.
(3) $d^{2}=0$.

These rules are extremely useful in computations.
We will make frequent use of the following lemma, which relates the exterior derivative of a one-form to the Lie bracket between two vector fields.

Lemma 2. Let $\omega \in \Omega^{1}(M)$ and let $X$ and $Y$ be smooth vector fields on $M$. Then

$$
d w(X, Y)=X \omega(Y)-Y \omega(X)-\omega([X, Y])
$$

Proof. It suffices to show that the lemma is true for a basis element, and hence for $\omega=f d g$. On the one hand, we have

$$
\begin{aligned}
d \omega(X, Y) & =(d f \wedge d g)(X, Y) \\
& =d f(X) d g(Y)-d f(Y) d g(X) \\
& =X(f) Y(g)-Y(f) X(g) .
\end{aligned}
$$

Furthermore,

$$
\begin{aligned}
X \omega(Y) & -Y \omega(X)-\omega([X, Y]) \\
& =X(f Y(g))-Y(f X(g))-f(X Y(g)-Y X(g)) \\
& =X(f) Y(g)-Y(f) X(g)
\end{aligned}
$$

and the lemma is proved.
This lemma gives the following version of Frobenius's theorem.
Theorem 3 (Frobenius). Let $\Delta$ be a $C^{\infty}$ distribution of dimension $k$ on $M$, an $n$-dimensional manifold. $\Delta$ is involutive if and only if there exist $n-k$ linearly independent one-forms $\omega^{k+1}, \ldots, \omega^{n}$ which vanish on $\Delta$ and satisfy

$$
\begin{equation*}
d \omega^{i}=\sum_{j=k+1}^{n} \theta_{j}^{i} \wedge \omega^{j} \quad i=k+1, \ldots, n \tag{2}
\end{equation*}
$$

for some set of one-forms $\theta_{j}^{\top}$.
Proof. The proof follows from application of Lemma 2 and Frobenius' theorem for vector fields. Let $X, Y$ be two vector fields in $\Delta$. Then

$$
[X, Y] \in \Delta \Longleftrightarrow \omega^{i}([X, Y])=0 \quad i=k+1, \ldots, n
$$

since the $\omega$ 's annihilate $\Delta$. Now applying Lemma 2 we have

$$
\begin{aligned}
{[X, Y] \in \Delta } & \Longleftrightarrow-d \omega^{i}(X, Y)+X \omega^{i}(Y)-Y \omega^{i}(X)=0 \\
& \Longleftrightarrow d \omega^{i}(X, Y)=0 \quad i=k+1, \ldots, n .
\end{aligned}
$$

It follows that $d \omega^{i}$ must have the form in equation (2) since $d \omega$ is annihilated on all vectors $X, Y \in \Delta$ and $\left\{\omega^{k+1}, \ldots, \omega^{n}\right\}$ form a basis for the space of one-forms which annihilate $\Delta$.
2.3. Pfaffian Exterior Differential Systems. Formally, an exterior differential system is given by an ideal $\mathcal{I} \subset \Omega(M)$ that is closed under exterior differentiation. Recall that an ideal $I$ satisfies

$$
\alpha \in \mathcal{I}, \beta \in \Omega(M) \quad \Longrightarrow \quad \alpha \wedge \beta \in \mathcal{I} .
$$



We will be primarily interested in the special case of exterior differential systems which are generated by a set of nonholonomic constraints and we focus on that case here.

A Pfaffian system is an exterior differential system which is generated by a set of linearly independent one-forms. Let $I$ be a codistribution spanned by a set of linear independent one-forms $\left\{\alpha^{i}\right\}, i=1, \ldots, s$. The ideal generated by $I$ is

$$
\mathcal{I}=\{I\}=\left\{\sigma \in \Omega: \sigma \wedge \alpha^{1} \cdots \wedge \alpha^{s}=0\right\}
$$

For an ideal generated by a set of one-forms, each element in the ideal has the form

$$
\xi=\sum_{j=1}^{s} a_{i j} \theta^{j} \wedge \alpha^{j}
$$

for some $\theta^{j} \in \Omega$.
It is also possible to rephrase Frobenius's Theorem in a concise way using ideals. Let $\mathcal{I}$ be the ideal generated by $\left\{\alpha^{1}, \ldots, \alpha^{0}\right\}$ and write $d \mathcal{I}$ for the set consisting of the exterior derivative of all elements of $\mathcal{I}$. We say that $\mathcal{I}$ is integrable if there exist functions $h_{1}, \ldots, h_{s}$ such that $\mathcal{I}=\left\{d h_{1}, \ldots, d h_{s}\right\}$. The Frobenius theorem asserts that the following set of relationships hold:

$$
\begin{align*}
\mathcal{I} \text { is integrable } & \Longleftrightarrow d \mathcal{I} \subset \mathcal{I} \\
& \Longleftrightarrow d \alpha^{i} \wedge \alpha^{1} \wedge \cdots \wedge \alpha^{s}=0 \\
& \Longleftrightarrow d \alpha^{i}=\sum_{j=1}^{s} \theta_{j}^{i} \wedge \alpha^{j} \quad \text { for some } \theta_{j}^{i}, i=1, \ldots, s  \tag{3}\\
& \Longleftrightarrow d \alpha^{i} \equiv 0 \bmod \mathcal{I} .
\end{align*}
$$

The last relationship in equation (3) uses the notion of congruence. Given two forms $\omega, \xi \in \Omega$, we write $\omega \equiv \xi \bmod \mathcal{I}$ if there exists an exterior form $\eta \in \mathcal{I}$ such that $\omega=\xi+\eta$. If $I$ is a set of one-forms (and hence not an ideal) then we write $\omega \equiv \xi \bmod I$ if there exist exterior forms $\alpha \in I$ and $\eta \in \Omega$ such that $\omega=\xi+\eta \wedge \alpha$. It follows that if $I$ is the generator set for an ideal $I$, then $\omega \bmod I=\omega \bmod I$. In the case that $I$ is generated by one-forms $\left\{\alpha_{i}\right\}$, we will often make use of the relationship

$$
\omega \bmod \mathcal{I} \equiv 0 \Longleftrightarrow \omega=\sum \theta_{i} \wedge \alpha^{i} \text { for some } \theta_{i} \in \Omega
$$

Although conceptually simple, mod-ing out by a set of one-forms can sometimes require considerable effort. For example, given the expression

$$
d x_{1} \wedge d x_{2}+d x_{2} \wedge d x_{3} \quad \bmod d x_{3}
$$

it is easy to see that this is congruent to $d x_{1} \wedge d x_{2}$. One merely sets all components containing a factor of $d x_{3}$ to zero. However, for the expression

$$
\left(d x_{1}+d x_{3}\right) \wedge d x_{2} \quad \bmod d x_{2}+d x_{3}
$$

this simple prescription will not work. The one-form must be rewritten in terms of an appropriate basis. For example, the above expression can be rewritten as

$$
\left(d x_{1}+d x_{3}\right) \wedge\left(d x_{2}+d x_{3}\right)-d x_{1} \wedge d x_{3} \bmod d x_{2}+d x_{3}
$$

and this is clearly congruent to $-d x_{1} \wedge d x_{3}$. More generally, to compute $\omega \bmod$ $\alpha^{1}, \ldots, \alpha^{s}$, one must rewrite $\omega$ in terms of a basis which includes the $\alpha^{i}$ 's as elements.
2.4. The derived flag. Let $I=\operatorname{span}\left\{\omega^{1}, \ldots, \omega^{s}\right\}$ be a smooth codistribution on $M$. The exterior derivative induces a mapping $\delta: I \rightarrow \Omega^{2}(M) / I$ :

$$
\delta: \lambda \mapsto d \lambda \bmod I \in \Omega^{2}(M) .
$$

The mapping $\delta$ is a linear mapping over $C^{\infty}(M)$ :

$$
\begin{aligned}
\delta(f \alpha+g \beta) & =d f \wedge \alpha+f d \alpha+d g \wedge \beta+g d \beta \bmod I \\
& =f d \alpha+g d \beta \bmod I \\
& =f \delta(\alpha)+g \delta(\beta) .
\end{aligned}
$$

It follows that the kernel of $\delta$ is a codistribution on $M$ (i.e. at each point $p \in M$, the kernel of $\delta$ is a linear subspace of $\left.T_{p}^{*} M\right)$. We call this subspace $I^{(1)}$, the first derived system of $I$ :

$$
I^{(1)}=\operatorname{ker} \delta=\{\lambda \in I: d \lambda \bmod I \equiv 0\}
$$

We can represent $I^{(1)}$ using a set of one-forms, but it is important to note that the basis for $I^{(1)}$ may not be a simple subset of the basis for $I$. Linear combinations of basis elements (over the ring of smooth functions on $M$ ) must be searched to find a basis for the derived system.

Since $I^{(1)}$ is itself a Pfaffian system, we can continue this construction and generate a nested sequence of codistributions

$$
\begin{equation*}
I=I^{(0)} \supset I^{(1)} \supset \cdots \supset I^{(N)} \tag{4}
\end{equation*}
$$

If the dimension of each $I^{(i)}$ is constant, then this construction terminates for some finite integer $N$. In this case, we call equation (4) the derived flag of $I$ and $N$ the derived length.

The derived flag describes the integrability properties of the ideal generated by $I$. If $I$ is completely integrable, then by Frobenius's theorem we have $I^{(1)}=I^{(0)}$, i.e. the length of the derived flag is zero. In fact, $I^{(N)}$ is always integrable since by definition $d I^{(N)} \bmod I^{(N)} \equiv 0 . I^{(N)}$ is the largest integrable subsystem contained in $I$. Thus if $I^{(N)}$ is not empty, then there exist functions $h_{1}, \ldots, h_{r}$ such that $\left\{d h_{i}\right\} \subset\{I\}$. In the context of control theory, this means that the system is not controllable since there exist algebraic functions which provide a foliation of the


Figure 1. Penny rolling on a plane. The state space is defined by the $\left(x_{1}, y_{1}\right)$ position of the penny, $x_{3}$, the angle of the penny with respect to the reference frame, and $x_{4}$ the amount the penny is rotated with respect to the vertical.
state space and it is impossible to move from one leaf of the foliation to another. The converse of this controllability result is provided by Chow's Theorem.

Theorem 4 (Chow). Let $I=\left\{\alpha^{1}, \ldots, \alpha^{s}\right\}$ represent a set of constraints and assumed that the derived flag of the system exists. Then, there exists a path $x(t)$ between any two points satisfying $\alpha^{i}(x) \dot{x}=0$ for all $i$ if and only if there exists an $N$ such that $I^{(N)}=\{0\}$.

Example 1. Consider the kinematic model of a penny rolling on a plane, as shown in Figure 1. Let $x \in \mathbb{R}^{4}$ denote the configuration of the penny, with $\left(x_{1}, x_{2}\right)$ being the location of the penny on the plane, $x_{3}$ the angle that the penny makes with a fixed line on the plane, and $x_{4}$ the angle of a fixed radial line on the penny with respect to the vertical. We take the radius of the penny as 1 . The constraints for the penny are that it roll in the direction it is pointing, with no slipping:

$$
\begin{align*}
& \alpha^{1}=\cos x_{3} d x_{1}+\sin x_{3} d x_{2}-d x_{4}  \tag{5}\\
& \alpha^{2}=\sin x_{3} d x_{1}-\cos x_{3} d x_{2} . \tag{6}
\end{align*}
$$

The exterior derivatives of $\alpha^{1}$ and $\alpha^{2}$ are given by

$$
\begin{align*}
& d \alpha^{1}=-\sin x_{3} d x_{3} \wedge d x_{1}+\cos x_{3} d x_{3} \wedge d x_{2}  \tag{7}\\
& d \alpha^{2}=\cos x_{3} d x_{3} \wedge d x_{1}+\sin x_{3} d x_{3} \wedge d x_{2} . \tag{8}
\end{align*}
$$

It is easy to check that the following relationships hold

$$
\begin{gathered}
d \alpha^{1} \wedge \alpha^{1}=-d x_{1} \wedge d x_{2} \wedge d x_{3}+\cos x_{3} d x_{2} \wedge d x_{3} \wedge d x_{4}-\sin x_{3} d x_{1} \wedge d x_{3} \wedge d x_{4} \\
d \alpha^{1} \wedge \alpha^{1} \wedge \alpha^{2}=0 \\
d \alpha^{2} \wedge \alpha^{1} \wedge \alpha^{2}=-d x_{1} \wedge d x_{2} \wedge d x_{3} \wedge d x_{4} \neq 0
\end{gathered}
$$

and hence the derived flag has the form:

$$
\begin{align*}
& I^{(0)}=\left\{\alpha^{1}, \alpha^{2}\right\} \\
& I^{(1)}=\left\{\alpha^{1}\right\}  \tag{9}\\
& I^{(2)}=\{0\} .
\end{align*}
$$

The dimension of the derived flag is $(2,1,0)$, implying that the system is completely controllable.

In control theory, Chow's theorem is usually stated by asking that the involutive closure of the distribution $I^{\perp}$ span the tangent space at each point $x \in M$. The connection between the Lie algebra formulation of Chow's theorem and the exterior differential system formulation is made with the following lemma.

Lemma 5. If $I=\Delta^{\perp}$ then $I^{(1)}=(\Delta+[\Delta, \Delta])^{\perp}$.
Proof. Follows from Lemma 2.
This lemma allows us to compute the derived flag for a system given the distribution $\Delta=I^{\perp}$. Define the nested set of distributions $E_{0} \subset E_{1} \subset \cdots \subset E_{N}$ as

$$
\begin{aligned}
& E_{0}=\Delta \\
& E_{i}=E_{i-1}+\left[E_{i-1}, E_{i-1}\right] .
\end{aligned}
$$

This sequence terminates if the dimension of each $E_{i}$ is constant, and it follows from Lemma 5 that $I^{(i)}=E_{i}^{\perp}$.

Remark 1. When doing computations with exterior differential systems, it is convenient to choose a basis of one-forms whose structure matches that of the derived flag. We say that a basis $\left\{\alpha^{i}\right\}$ is adapted to the derived flag if

$$
I^{(i)}=\left\{\alpha^{1}, \ldots, \alpha^{s_{i}}\right\}
$$

where $s_{i}$ is a strictly decreasing sequence of integers. In other words, an adapted basis is one in which the derived systems are calculated by dropping elements from the end of the basis. An adapted basis can be calculated by computing the derived flag and then choosing the basis elements starting with a basis for $I^{(N-1)}$ and proceeding backwards.
2.5. Pfaff's problem and Engel's theorem. The simplest type of normal form for a nonholonomic system involves finding a normal form for a single constraint.

Theorem 6 (Pfaff's problem). Suppose $\alpha$ is a one-form which satisfies $(d \alpha)^{r+1} \wedge$ $\alpha=0,(d \alpha)^{r} \wedge \alpha \neq 0$. Then there exist coordinates such that

$$
\alpha=d x_{1}+x_{2} d x_{3}+\cdots+x_{2 r} d x_{2 r+1}
$$

The proof uses a number of tools that are beyond the scope of this paper. In the $r=1$ case, the proof reduces to proving that there exist two functions $f_{1}$ and $f_{2}$ which satisfy

$$
\begin{align*}
& d \alpha \wedge \alpha \wedge d f_{1}=0  \tag{10}\\
& \alpha \wedge d f_{1} \wedge d f_{2}=0
\end{align*} \quad \text { and } \quad \alpha \wedge d f_{1} \neq 0,
$$

Given $f_{1}$ and $f_{2}, \alpha$ can be scaled such that

$$
\alpha=d f_{2}+g d f_{1}=: d x_{1}+x_{2} d x_{3}
$$

The Pfaff theorem guarantees that these equations have a solution (it need not be unique).

In the case of a single constraint in $\mathbb{R}^{3}$, Pfaff's theorem shows that if the constraint is non-integrable then the corresponding control system can be written in chained form. This follows because if $\alpha$ is not integrable then $d \alpha \wedge \alpha \neq 0$ but $(d \alpha)^{2} \wedge \alpha=0$ by a dimension count. Therefore, we can apply Theorem 6 (with a relabeling of coordinates) to conclude that

$$
\alpha=d \xi_{3}-\xi_{2} d \xi_{1}
$$

A basis for the right null space of this constraint is then given by

$$
g_{1}=\frac{\partial}{\partial \xi_{1}}+\xi_{2} \frac{\partial}{\partial \xi_{3}} \quad g_{2}=\frac{\partial}{\partial \xi_{2}}
$$

which is the chained form for two input vector fields in $\mathbb{R}^{3}$.
Engel's theorem applies to the case of two non-integrable constraints in $\mathbb{R}^{4}$.
Theorem 7 (Engel's theorem). Let I be a two-dimensional codistribution on $\mathbb{R}^{4}$ with $\operatorname{dim} I^{(1)}=1$ and $\operatorname{dim} I^{(2)}=0$. Then there exist local coordinates such that

$$
\begin{equation*}
I=\left\{d \xi_{4}-\xi_{3} d \xi_{1}, d \xi_{3}-\xi_{2} d \xi_{1}\right\} \tag{11}
\end{equation*}
$$

Proof. Choose a basis $I=\left\{\alpha^{1}, \alpha^{2}\right\}$ which is adapted to the derived flag. It follows that $d \alpha^{1} \wedge \alpha^{1} \neq 0$ and $\left(d \alpha^{1}\right)^{2} \wedge \alpha^{1}=0$ (by dimension count). Hence we can use Pfaff's theorem to find coordinates such that $\alpha^{1}=d \xi_{4}-\xi_{3} d \xi_{1}$.

To determine $\xi_{2}$, we use the structure of $\alpha^{2}$. Since $\alpha^{1} \in I^{(1)}$, we have $d \alpha^{1} \wedge \alpha^{1} \wedge$ $\alpha^{2}=0$. But $d \alpha^{1}=-d \xi_{3} \wedge d \xi_{1}$ and hence

$$
\alpha^{2} \equiv a d \xi_{3}+b d \xi_{1} \quad \bmod \alpha^{1}
$$

Since $\alpha^{2} \neq 0$, it follows that we can not have both $a$ and $b=0$. We split the proof into two cases.
Case 1: $(a \neq 0)$. Since $\alpha^{2}$ is only determined $\bmod \alpha^{1}$, we are free to scale $\alpha^{2}$ by any nonzero function. Hence

$$
\begin{equation*}
\frac{1}{a} \alpha^{2} \equiv d \xi_{3}+\frac{b}{a} d \xi_{1} \bmod \alpha^{1} \tag{12}
\end{equation*}
$$

and choosing $\xi_{2}=-b / a$ yields a basis for the codistribution which is in Engel's normal form. Notice that the basis is a transformed version of the original basis, namely,

$$
\begin{array}{ll}
\bar{\alpha}_{1}=\alpha_{1} & =d \xi_{4}-\xi_{3} d \xi_{1} \\
\bar{\alpha}_{2}=\frac{1}{a} \alpha_{2}+\lambda \alpha_{1} & =d \xi_{3}-\xi_{2} d \xi_{1}
\end{array}
$$

where $\lambda$ is chosen such that equation (12) becomes an equality.
Case 2: $(b \neq 0)$. In this case we can scale $\alpha^{2}$ so that

$$
\frac{1}{b} \alpha^{2} \equiv \frac{a}{b} d \xi_{3}+d \xi_{1} \quad \bmod \alpha_{1}
$$

Defining $\xi_{2}=-a / b$ gives the normal form

$$
\begin{aligned}
& \bar{\alpha}_{1}=d \xi_{4}-\xi_{3} d \xi_{1} \\
& \bar{\alpha}_{2}=d \xi_{1}-\xi_{2} d \xi_{3} .
\end{aligned}
$$

It turns out that this normal form is diffeomorphic to Goursat form via the following change of coordinates:

$$
\begin{aligned}
& \eta_{1}=\xi_{3} \\
& \eta_{2}=-\xi_{2} \\
& \eta_{3}=-\xi_{1} \\
& \eta_{4}=\xi_{4}-\xi_{1} \xi_{3}
\end{aligned} \quad \Rightarrow \quad \begin{aligned}
& \\
& \alpha^{1}=d \eta_{4}-\eta_{3} d \eta_{1} \\
& \alpha^{2}=d \eta_{3}-\eta_{2} d \eta_{1}
\end{aligned}
$$

Hence the transformed basis is in Engel's normal form.
2.6. Goursat normal form. We now turn to the more general case of $n-2$ constraints on an $n$-dimensional manifold $M$. Let $I$ be a codistribution on $M$ whose derived flag satisfies $\operatorname{dim} I^{(i)}=n-i-2$.
Theorem 8 (Goursat normal form). Let $U$ be an open subset of $\mathbb{R}^{n}$ and $I=$ $\left\{\alpha^{1}, \ldots, \alpha^{s}\right\}$ be a collection of $s=n-2$ smooth, linearly independent one-forms defined on $U$. If there exists a one-form $\pi \neq 0 \bmod I$ such that

$$
\begin{align*}
& d \alpha^{i} \equiv-\alpha^{i+1} \wedge \pi \bmod \alpha^{1}, \ldots, \alpha^{i} \quad i=1, \ldots, s-1 \\
& d \alpha^{s} \neq 0 \bmod I \tag{13}
\end{align*}
$$

then there exists a set of coordinates $\xi$ such that

$$
I=\left\{d \xi_{n}-\xi_{n-1} d \xi_{1}, \ldots, d \xi_{3}-\xi_{2} d \xi_{1}\right\}
$$

A few comments on the statement of this theorem are in order. The conditions of the theorem require the existence of a special basis $\left\{\alpha^{i}\right\}$ and a special one-form $\pi$. A quick calculation shows that the basis $\left\{\alpha^{i}\right\}$ is adapted to the derived flag of the system and hence if we start with an adapted basis, the real requirement is the existence of a one-form $\pi$ which satisfies the congruences. Determining $\pi$ can involve a further scaling of the adapted basis which preserves the adapted structure
(see Section 3.2 for an example). For most examples, $\pi$ can be determined by a combination of physical insight and repeated guessing.

A complete proof of this theorem can be found in [4]. It can be summarized in the following algorithm for converting a system into Goursat form (see [12] for the feedback linearization version of this algorithm, on which this is based).
Algorithm 1. Given a codistribution $I=\left\{\omega^{1}, \ldots, \omega^{s}\right\}$ with $s=n-2$, the following steps are required:
(1) Construct a basis $I=\left\{\alpha^{1}, \ldots, \alpha^{s}\right\}$ which is adapted to the derived flag. Check the Goursat congruences to ensure they are satisfied for some $\pi$.
(2) It follows from the congruences that $\alpha^{1}$ and $\alpha^{2}$ satisfy $d \alpha^{1} \wedge \alpha^{1} \wedge \alpha^{2}=0$ and hence the proof of Engel's theorem can be used to find coordinates such that

$$
\begin{aligned}
& \alpha^{1}=d \xi_{n}-\xi_{n-1} d \xi_{1} \\
& \alpha^{2}=d \xi_{n-1}-\xi_{n-2} d \xi_{1} .
\end{aligned}
$$

This may involve finding a new basis which preserves the adapted structure, as well as a change of coordinates, to convert between the two normal forms in the proof of Engel's theorem.
(3) The remaining coordinates are determined by simple differentiation. Given $\xi_{i}$ we determine $\xi_{i-1}$ by algebraically solving the equation

$$
\alpha^{n-i+1} \equiv d \xi_{i}+\xi_{i-1} d \xi_{1} \quad \bmod \alpha^{1}, \ldots, \alpha^{n-i+1}
$$

The proof of Goursat's theorem is to essentially show that this equation always has a solution.

Example 2. Consider again the rolling penny from the previous example. The basis

$$
I=\left\{\alpha^{1}, \alpha^{2}\right\} \quad \begin{array}{ll}
\alpha^{1}=\cos x_{3} d x_{1}+\sin x_{3} d x_{2}-d x_{4} \\
& \alpha^{2}=\sin x_{3} d x_{1}-\cos x_{3} d x_{2}
\end{array}
$$

was shown to be a basis adapted to the derived flag.
Next, we search for a one-form $\pi$ which satisfies the Goursat congruences. Since this system is low dimensional, the algebra needed to find $\pi$ is straightforward. Define $\alpha^{3}$ and $\alpha^{4}$ so as to complete the basis:

$$
\begin{aligned}
& \alpha^{3}=\cos x_{3} d x_{1}+\sin x_{3} d x_{2} \\
& \alpha^{4}=d x_{3} .
\end{aligned}
$$

$\pi$ must satisfy

$$
\begin{equation*}
d \alpha^{1}=-\alpha^{2} \wedge \pi \bmod \alpha^{1} ; \tag{14}
\end{equation*}
$$

Setting $\pi=\lambda_{3} \alpha^{3}+\lambda_{4} \alpha^{4}$, equation (14) gives

$$
\begin{aligned}
& \lambda_{3}=0 \\
& \lambda_{4}=1 \quad \Longrightarrow \quad \pi=d x_{3} .
\end{aligned}
$$

We now proceed to apply the algorithm for converting the system to Goursat form. In this case, only the Engel's step is required. We begin by solving Pfaff's problem. The first partial differential equation to be solved is $d \alpha^{1} \wedge \alpha^{1} \wedge d f_{1}=0$, which yields

$$
\left(-d x_{1} \wedge d x_{2} \wedge d x_{3}-\sin x_{3} d x_{1} \wedge d x_{3} \wedge d x_{4}+\cos x_{3} d x_{2} \wedge d x_{3} \wedge d x_{4}\right) \wedge d f_{1}=0
$$

This equation has a trivial solution given by $f_{1}=x_{3}$. The second partial differential equation, $\alpha^{1} \wedge d f_{1} \wedge d f_{2}=0$, then becomes

$$
\left(\cos x_{3} d x_{1} \wedge d x_{3}+\sin x_{3} d x_{2} \wedge d x_{3}+d x_{3} \wedge d x_{4}\right) \wedge d f_{2}=0
$$

which has a solution given by $f_{2}=x_{1} \cos x_{3}+x_{2} \sin x_{3}-x_{4}$.
Finally, we solve for the remaining coordinate by finding functions $a$ and $b$ such that

$$
\alpha^{1}=a d f_{1}+b d f_{2}
$$

This is a completely algebraic problem which has a solution given by

$$
\begin{aligned}
a & =x_{1} \sin x_{3}-x_{2} \cos x_{3} \\
b & =1 .
\end{aligned}
$$

Combining all of these calculations, we define

$$
\begin{aligned}
& \xi_{1}=x_{3} \\
& \xi_{3}=-x_{1} \sin x_{3}+x_{2} \cos x_{3} \\
& \xi_{4}=x_{1} \cos x_{3}+x_{2} \sin x_{3}-x_{4}
\end{aligned}
$$

which gives $\alpha^{1}=d \xi_{4}-\xi_{3} d \xi_{1}$.
We now define $\xi_{2}$ by examining $\alpha^{2}$. From the proof of Engel's theorem we must have

$$
\alpha^{2} \equiv a d \xi_{3}+b d \xi_{1} \quad \bmod \alpha^{1}
$$

Performing all calculations in the original coordinates, this relationship becomes

$$
\begin{aligned}
& \sin x_{3} d x_{1}-\cos x_{3} d x_{2} \equiv \\
& \quad a\left(-\sin x_{3} d x_{1}+\cos x_{3} d x_{2}-\left(x_{1} \cos x_{3}+x_{2} \sin x_{3}\right) d x_{3}\right)+b d x_{3} \bmod \alpha^{1}
\end{aligned}
$$

from which it follows that $a=-1$ and $b=-\left(x_{1} \cos x_{3}+x_{2} \sin x_{3}\right)$. This is case 1 in the proof of Engel's theorem and hence

$$
\xi_{2}=b / a=\left(x_{1} \cos x_{3}+x_{2} \sin x_{3}\right)
$$

and $\bar{\alpha}^{2}=-\alpha^{2}$ is the new basis element. This completes the change of basis and change of coordinates, and the resulting system is in Goursat normal form.

A more complex example, the $N$-trailer system, is the subject of Section 3.
2.7. Converting systems to chained form. Chained form is dual to the Goursat normal form presented above. That is, a system with constraints in Goursat normal form can always be written as a control system in chained form by choosing


$$
\begin{aligned}
& g_{1}=\frac{\partial}{\partial x_{1}}+x_{2} \frac{\partial}{\partial x_{3}}+\cdots+x_{n-1} \frac{\partial}{\partial x_{n}} \\
& g_{2}=\frac{\partial}{\partial x_{2}}
\end{aligned}
$$

which forms a basis for the distribution annihilated by $I$. Thus, we can formulate the problem of finding a basis for the constraints which is in Goursat form as the problem of finding a feedback transformation to convert a system to chained form

The Goursat congruences are somewhat unsatisfying since they require the existence of a one-form $\pi$. Necessary and sufficient conditions for the existence of such a $\pi$, and hence for converting a set of constraints into Goursat normal form, were presented in [27]. We summarize the main result here.

Let $I=\operatorname{span}\left\{\omega^{1}, \ldots, \omega^{6}\right\}$ be a codistribution on $\mathbb{R}^{n}$ and write $\Delta=I^{\perp}$ for the distribution which spans the null space of the codistribution. We define two nested sets of distributions:

$$
\begin{array}{cc}
E_{0}=\Delta & F_{0}=\Delta \\
E_{1}=E_{0}+\left[E_{0}, E_{0}\right] & F_{1}=F_{0}+\left[F_{0}, F_{0}\right] \\
E_{2}=E_{1}+\left[E_{1}, E_{1}\right] & F_{2}=F_{1}+\left[F_{1}, F_{0}\right]  \tag{15}\\
& \vdots \\
& \vdots \\
E_{i+1} & =E_{i}+\left[E_{i}, E_{i}\right]
\end{array} F_{i+1}=F_{i}+\left[F_{i}, F_{0}\right] .
$$

Under the assumption that each distribution is constant rank, the two sequences have finite length (possibly different).

The filtration $\left\{F_{i}\right\}$ is the the one which usually appears in the context of nonlinear controllability and feedback linearization. In particular, $F_{i}$ consists of all brackets up to order $i$. The distribution $E_{i}$ also contains all brackets of order $i$, but may contain additional Lie products of higher order. This is due to the recursive construction of $E_{i}$, as opposed to the iterative construction of $F_{i}$. The filtration $E_{i}$ is precisely the sequence of distributions which is perpendicular to the derived flag of $I=\Delta^{\perp}$.
Theorem 9 ([27]). Given a 2-dimensional distribution $\Delta=I^{\perp}$ such that

$$
\operatorname{dim} E_{i}=\operatorname{dim} F_{i}=i+2 \quad i=0, \ldots, n-2,
$$

there exists a basis $\left\{\alpha^{1}, \ldots, \alpha^{s}\right\}$ for I which is in Goursat normal form.
This theorem allows us to completely characterize the set of systems which are equivalent to a system in chained (or Goursat) form in the case that the relative growth vector of the system is $\sigma=(2,1, \ldots, 1)$. It can be shown that the $N$-trailer problem satisfies the conditions of Theorem 9 and hence it can be converted to chained form.


Figure 2. The mobile robot Hilare with $n$ trailers.

## 3. The Motion Planning Problem for the N-trailer system

In this section, we define the Pfaffian system (set of one-forms which represent the velocity constraints) for the $N$-trailer problem and calculate its derived flag. We then show how the system can be converted into either Goursat normal form (following Theorem 8 and Algorithm 1) or its dual, chained form. Although the calculations in this section assume a particular configuration of the mobile robot and trailer system, we will show that our model is general enough to encompass not only the specific choice we have made but also a front-wheel drive car pulling trailers and the luggage trains found in airports.
3.1. The system of rolling constraints and its derived flag. Consider a mobile robot such as Hilare ${ }^{1}$ with $n$ trailers attached, as in Figure 2. Each trailer is attached to the body in front of it by a rigid bar, and the rear set of wheels of each body is constrained to roll without slipping. The trailers are assumed to be identical, but to have possibly different link lengths $L_{i}$. The $x, y$ coordinates of a midpoint between the two wheels are referred to as $\left(x_{i}, y_{i}\right)$ and the hitch angles (all measured with respect to the horizontal) are $\theta_{i}$. The connections between the bodies give rise to the following constraints:

$$
\begin{align*}
& x_{i}=x_{i-1}-L_{i} \cos \theta_{i} \\
& y_{i}=y_{i-1}-L_{i} \sin \theta_{i}, \tag{16}
\end{align*}
$$

$i=1,2, \ldots, n$ for the general case with $n$ trailers. These constraints are holonomic and will reduce the dimension of the configuration space, since the positions $\left(x_{i}, y_{i}\right)$ for $i \geq 1$ can be expressed in terms of $x_{0}, y_{0}, \theta_{0}, \ldots, \theta_{i}$. By symmetry, $\left(x_{i}, y_{i}\right)$ for $i<n$ can also be expressed in terms of $x_{n}, y_{n}, \theta_{n}, \theta_{n-1}, \ldots, \theta_{i}$. For our purposes it will be far more useful to use as configuration space variables the $x, y$ coordinates of a point on the $n^{\text {th }}$ trailer and the $n+1$ hitch angles: $x_{n}, y_{n}, \theta_{n}, \ldots, \theta_{0}$ because the calculations that follow are vastly simplified. ${ }^{2}$ We will refer to the state space as $x=\left(x_{n}, y_{n}, \theta_{n}, \ldots, \theta_{1}, \theta_{0}\right)$. We have assumed that the bodies are connected between

[^1]the midpoints of the two sets of rear wheels; it should be noted that if the trailers are hitched behind the rear axle, the equations will not simplify as shown here.

The wheels of the robot and trailers are constrained to roll without slipping; this implies that the velocity of each body in the direction perpendicular to its wheels must be zero. We model each pair of rear wheels as a single wheel at the midpoint of the axle, and state the non-slipping conditions in terms of coordinates, beginning with the $\boldsymbol{n}^{\text {th }}$ trailer:

$$
\begin{equation*}
0=\dot{x}_{n} \sin \theta_{n}-\dot{y}_{n} \cos \theta_{n} . \tag{17}
\end{equation*}
$$

Equation (17) models the fact that the velocity perpendicular to the wheels is zero. In the language of one forms we write this as

$$
\begin{equation*}
\alpha^{1}\left(x_{n}, y_{n}, \theta_{n}, \ldots, \theta_{0}\right)=\sin \theta_{n} d x_{n}-\cos \theta_{n} d y_{n} \tag{18}
\end{equation*}
$$

To write the other rolling constraints, we define $v_{i}$ to be the magnitude of the velocity of the $i^{\text {th }}$ trailer. The direction of motion of the $(i+1)^{s t}$ trailer and consequently the direction of $v_{i+1}$, if its wheels are rolling without slipping, is along the direction of the hitch joining the $(i+1)^{s t}$ body to the $i^{\text {th }}$ body. Since the bodies are linked together by rigid rods, it follows that the projection of $v_{i}$ onto the line of the hitch is equal to $v_{i+1}$. Thus, we have that

$$
\begin{equation*}
v_{i+1}(x)=\cos \left(\theta_{i+1}-\theta_{i}\right) v_{i}(x) \tag{19}
\end{equation*}
$$

Also, we have that the velocity of the $n^{\text {th }}$ trailer $v_{n}$ is given by

$$
\begin{equation*}
v_{n}(x)=\cos \theta_{n} \dot{x}_{n}+\sin \theta_{n} \dot{y}_{n} \tag{20}
\end{equation*}
$$

In the sequel we will need to use $v_{n}$ as a one form (i.e. we will need to use $v_{n} d t$ ) and we denote this by abuse of notation as:

$$
\begin{equation*}
v_{n}(x)=\cos \theta_{n} d x_{n}+\sin \theta_{n} d y_{n} \tag{21}
\end{equation*}
$$

We may now recursively write down the rolling without slipping constraints for all the trailers. The velocity of each trailer has a component due to the velocity $v_{i+1}$ of the previous trailer and a component $L_{i+1} \dot{\theta}_{i+1}$ due to the rotation of the hitch. The relative geometry of this situation is illustrated in Figure 3. The component of $v_{i+1}$ in the direction perpendicular to the wheel base is $v_{i+1} \sin \left(\theta_{i}-\theta_{i+1}\right)$ and the component of $L_{i+1} \dot{\theta}_{i+1}$ in this direction is $L_{i+1} \dot{\theta}_{i+1} \cos \left(\theta_{i}-\theta_{i+1}\right)$. If the $i^{\text {th }}$ trailer rolls without slipping then we must have

$$
\begin{equation*}
0=L_{i+1} \dot{\theta}_{i+1} \cos \left(\theta_{i+1}-\theta_{i}\right)-v_{i+1} \sin \left(\theta_{i+1}-\theta_{i}\right) . \tag{22}
\end{equation*}
$$

Dividing through the equation (22) by $\cos \left(\theta_{i+1}-\theta_{i}\right)$ yields the form constraint for $n-1 \geq i \geq 0$ :

$$
\begin{equation*}
\alpha^{n+1-i}(x)=L_{i+1} d \theta_{i+1}-\tan \left(\theta_{i+1}-\theta_{i}\right) v_{i+1}=0 \tag{23}
\end{equation*}
$$

Note that we have used the one form version of $v_{i+1}$ in equation (23).
The forms $\alpha^{1}(x), \alpha^{2}(x), \ldots, \alpha^{n+1}(x)$ represent the constraints that the wheels of the $n^{t h},(n-1)^{s t}, \ldots, 0^{t h}$ trailer (i.e. the cab), respectively roll without slipping.


Figure 3. Showing the definitions of the angles and velocities of the $i^{\text {th }}$ trailer

They are given by the formulas (23) with the recursion relations (19). Thus, the Pfaffian system for the $N$-trailer problem is:

$$
\begin{equation*}
I=\operatorname{span}\left\{\alpha^{1}, \alpha^{2}, \ldots, \alpha^{n+1}\right\} \tag{24}
\end{equation*}
$$

The following theorem gives the derived flag associated with this Pfaffian system.
Theorem 10 (Derived Flag for the $N$-trailer Pfaffian system). Consider the Pfaffian system of the $N$-trailer system (24) with the one forms $\alpha^{i}$ defined by equations (18) and (23). The one-forms $\alpha^{i}$ are adapted to the derived flag in the following sense:

$$
\begin{align*}
I^{(0)} & =\operatorname{span}\left\{\alpha^{1}, \alpha^{2}, \ldots, \alpha^{n}, \alpha^{n+1}\right\} \\
I^{(1)} & =\operatorname{span}\left\{\alpha^{1}, \alpha^{2}, \ldots, \alpha^{n}\right\} \\
& \vdots  \tag{25}\\
I^{(n)} & =\operatorname{span}\left\{\alpha^{1}\right\} \\
I^{(n+1)} & =\{0\} .
\end{align*}
$$

Proof. The proof is by recursion starting from the bottom of the flag of (25). Indeed for the first step, we compute $d \alpha^{1}$ to be $d\left(\sin \theta_{n} d x_{n}-\cos \theta_{n} d y_{n}\right)$, namely:

$$
\begin{aligned}
d \alpha^{1} & =\cos \theta_{n} d \theta_{n} \wedge d x_{n}+\sin \theta_{n} d \theta_{n} \wedge d y_{n} \\
& =\left(-\cos \theta_{n} d x_{n}-\sin \theta_{n} d y_{n}\right) \wedge d \theta_{n} \\
& =-v_{n} \wedge d \theta_{n} .
\end{aligned}
$$

From Equation (21) it follows that $d \alpha^{1} \neq 0 \bmod \alpha^{1}$. This establishes the last two steps of the derived flag above. For the preceding step, we note that the form $\alpha^{2}$ is given by

$$
\alpha^{2}=L_{n} d \theta_{n}-\tan \left(\theta_{n}-\theta_{n-1}\right) v_{n} .
$$

This yields that $d \theta_{n}$ is proportional to $v_{n} \bmod \alpha^{2}$. Consequently, we have that $d \alpha^{1}=-v_{n} \wedge d \theta_{n}$ is equal to $0 \bmod \alpha_{2}$. This establishes that

$$
\begin{align*}
I^{(n-1)} & =\operatorname{span}\left\{\alpha^{1}, \alpha^{2}\right\} \\
I^{(n)} & =\operatorname{span}\left\{\alpha^{1}\right\}  \tag{26}\\
I^{(n+1)} & =\{0\} .
\end{align*}
$$

For the $i^{\text {th }}$ step of the recursion proof, we assume that we have shown that

$$
\begin{align*}
I^{(n-i+2)} & =\operatorname{span}\left\{\alpha^{1}, \alpha^{2}, \ldots, \alpha^{i-1}\right\} \\
& \vdots  \tag{27}\\
I^{(n)} & =\operatorname{span}\left\{\alpha^{1}\right\} \\
I^{(n+1)} & =\{0\} .
\end{align*}
$$

We need to show that $d \alpha^{i}=0 \bmod \alpha^{1}, \ldots, \alpha^{i-1}, \alpha^{i}$. To verify this it is useful to have the following preliminary lemma:
Lemma 11. For the one forms $v_{i}$ we have that

$$
\begin{equation*}
d v_{n-i} \equiv 0 \bmod \alpha^{1}, \alpha^{2}, \ldots, \alpha^{i+2} \tag{28}
\end{equation*}
$$

Proof. Start first with $d v_{n}=d\left(\cos \theta_{n} d x_{n}+\sin \theta_{n} d y_{n}\right)$ :

$$
\begin{aligned}
d v_{n} & =-\sin \theta_{n} d \theta_{n} \wedge d x_{n}+\cos \theta_{n} d \theta_{n} \wedge d y_{n} \\
& =\left(\sin \theta_{n} d x_{n}-\cos \theta_{n} d y_{n}\right) \wedge d \theta_{n} \\
& \equiv 0 \bmod \alpha^{1} .
\end{aligned}
$$

Thus $d v_{n} \equiv 0 \bmod \alpha^{1}$.
From $v_{n-1}=v_{n} \sec \left(\theta_{n}-\theta_{n-1}\right)$ it follows that

$$
d v_{n-1}=d v_{n} \sec \left(\theta_{n}-\theta_{n-1}\right)+\sec \left(\theta_{n}-\theta_{n-1}\right) \tan \left(\theta_{n}-\theta_{n-1}\right) v_{n} \wedge\left(d \theta_{n}-d \theta_{n-1}\right)
$$

The first term is zero mod $\alpha^{1}$ since $d v_{n} \equiv 0 \bmod \alpha^{1}$. The second term is zero mod $\alpha^{2}$ since $v_{n}$ is proportional to $d \theta_{n} \bmod \alpha^{2}$ and the third term is zero $\bmod \alpha^{3}$ since $v_{n}$ is proportional to $\theta_{n-1} \bmod \alpha^{3}$. Thus, we have that

$$
d v_{n-1} \equiv 0 \quad \bmod \alpha^{1}, \alpha^{2}, \alpha^{3}
$$

Proceeding recursively, we have that

$$
d v_{n-i} \equiv 0 \quad \bmod \alpha^{1}, \alpha^{2}, \ldots, \alpha^{i+2}
$$

which completes the proof of the lemma.
We will also need to make use of the relations:

$$
\begin{align*}
d \theta_{n} & \equiv v_{n} & & \bmod \alpha^{2} \\
d \theta_{i} & \equiv v_{n} & & \bmod \alpha^{n-i+2}  \tag{29}\\
d \theta_{n-i+2} & \equiv v_{n} & & \bmod \alpha^{i} .
\end{align*}
$$

These follow directly from the definition of the $\alpha^{i}$ in Equation (23) and the linear dependence of the one-forms $v_{i}$, given in Equation (19).

Continuing with the proof of the theorem, we now begin the calculation for

$$
\begin{aligned}
d \alpha^{i}= & d\left(L_{n-i+2} d \theta_{n-i+2}-\tan \left(\theta_{n-i+2}-\theta_{n-i+1}\right) v_{n-i+2}\right) \\
= & -\sec ^{2}\left(\theta_{n-i+2}-\theta_{n-i+1}\right)\left(d \theta_{n-i+2}-d \theta_{n-i+1}\right) \wedge v_{n-i+2} \\
& -\tan \left(\theta_{n-i+2}-\theta_{n-i+1}\right) d v_{n-i+2} .
\end{aligned}
$$

This expression has three terms. By equation (28), we have that $d v_{n-i+2} \equiv 0$ $\bmod \alpha^{1}, \ldots, \alpha^{i}$. Also by the proportionality of the $d \theta_{i}$ to $v_{n}(29)$ and the linear dependence of the $v_{i}$ 's (19), we have that $d \theta_{n-i+2} \wedge v_{n-i+2} \equiv 0 \bmod \alpha^{i}$ and $d \theta_{n-i+1} \wedge$ $v_{n-i+2} \equiv 0 \bmod \alpha^{i-1}$. Thus, we have that $d \alpha^{i} \equiv 0 \bmod \alpha^{1}, \alpha^{2}, \ldots, \alpha^{i}$ which implies that the derived flag has the form

$$
I^{(n-i+1)}=\left\{\alpha^{1}, \ldots, \alpha^{i}\right\}
$$

as stated.
We note that the $I^{(n+1)}=\{0\}$ implies that the $N$-trailer system is completely controllable (by Chow's theorem).
3.2. Conversion to Goursat Normal Form. In the preceding subsection, we have shown that the ideal generated by $\alpha^{1}, \ldots, \alpha^{n+1}$ defined in equations (18) and (23) is adapted to its derived flag in the sense of (25). It remains to check whether the $\alpha^{i}$ satisfy the Goursat congruences and if they do, to find a transformation that puts them into the Goursat canonical form.

Theorem 12 (Goursat Congruences for the $N$-trailer system). Consider the Pfaffian system associated with the $N$-trailer system (24) with the one-forms $\alpha^{i}$ defined by equations (18) and (23), repeated below:

$$
\begin{aligned}
\alpha^{1}(x)= & \sin \theta_{n} d x_{n}-\cos \theta_{n} d y_{n} \\
\alpha^{i}(x)= & L_{n-i+2} d \theta_{n-i+2}-\tan \left(\theta_{n-i+2}-\theta_{n-i+1}\right) v_{n-i+2} \\
& \quad i=2, \ldots, n+1 .
\end{aligned}
$$

There exists a change of basis of the one forms $\alpha^{i}$ to $\bar{\alpha}^{i}$ which preserves the adapted structure, and a one-form $\pi$ which satisfies the Goursat congruences for this new basis:

$$
\begin{aligned}
d \bar{\alpha}^{i} & \equiv-\bar{\alpha}^{i+1} \wedge \pi \bmod \bar{\alpha}^{1}, \ldots, \bar{\alpha}^{i} \quad i=1, \ldots, n \\
d \bar{\alpha}^{n+1} & \neq 0 \bmod I .
\end{aligned}
$$

The one-form which satisfies these congruences is given by

$$
\pi=\cos \theta_{n} d x_{n}+\sin \theta_{n} d y_{n}
$$

and is equivalent to $v_{n}$, the velocity form of the $n^{\text {th }}$ trailer.
Proof. The outline for the proof is first to determine a suitable one-form $\pi$ from the first Goursat congruence, $d \alpha^{1} \equiv-\alpha^{2} \wedge \pi$. Then we construct the new basis elements $\bar{\alpha}^{i}$ one at a time such that they satisfy the rest of the congruences. For this example, we find that these new basis elements are multiples of the original
basis elements, and since the original basis is adapted to the derived flag, the new basis is also adapted.

We determine $\pi$ by following the same procedure as in Example 2. We complete the basis of $\left\{\alpha^{1}, \ldots, \alpha^{n+1}\right\}$ with

$$
\begin{aligned}
\alpha^{n+2} & =\cos \theta_{n} d x_{n}+\sin \theta_{n} d y_{n}=v_{n} \\
\alpha^{n+3} & =d \theta_{0}
\end{aligned}
$$

Note that $\alpha^{n+2}=v_{n}$, the velocity form of the last trailer. We then set $\pi=\lambda_{1} \alpha^{n+2}+$ $\lambda_{2} \alpha^{n+3}$ and solve for $\lambda_{1}, \lambda_{2}$ using

$$
d \alpha^{1} \equiv-\alpha^{2} \wedge \pi \quad \bmod \alpha^{1}
$$

Calculating the exterior derivative of $\alpha^{1}$,

$$
\begin{equation*}
d \alpha^{1}=\cos \theta_{n} d \theta_{n} \wedge d x_{n}+\sin \theta_{n} d \theta_{n} \wedge d y_{n}=d \theta_{n} \wedge v_{n} \tag{30}
\end{equation*}
$$

and then examining $\alpha^{2} \wedge \pi$,

$$
\alpha^{2} \wedge \pi=\left(L_{n} d \theta_{n}-\tan \left(\theta_{n}-\theta_{n-1}\right) v_{n}\right) \wedge\left(\lambda_{1} v_{n}+\lambda_{2} d \theta_{0}\right)
$$

we see that if we choose $\lambda_{1}=1, \lambda_{2}=0$, then

$$
\alpha^{2} \wedge \pi=L_{n} d \theta_{n} \wedge v_{n}=L_{n} d \alpha^{1}
$$

We note here that we could have chosen $\lambda_{1}=-1 / L_{n}$, but instead we will define a new basis element $\bar{\alpha}^{2}=-\left(1 / L_{n}\right) \alpha^{2}$. Then the one-form $\pi=v_{n}$ will satisfy

$$
d \alpha^{1}=-\bar{\alpha}^{2} \wedge \pi
$$

We now continue this procedure to find the rest of the transformed basis. Taking the exterior derivative of $\bar{\alpha}^{2}$,

$$
\begin{aligned}
d \bar{\alpha}^{2} & =d\left(-d \theta_{n}+\frac{1}{L_{n}} \tan \left(\theta_{n}-\theta_{n-1}\right) v_{n}\right) \\
& =\frac{1}{L_{n}} \sec ^{2}\left(\theta_{n}-\theta_{n-1}\right)\left(d \theta_{n}-d \theta_{n-1}\right) \wedge v_{n}-\frac{1}{L_{n}} \tan \left(\theta_{n}-\theta_{n-1}\right) d v_{n}
\end{aligned}
$$

and noting that

$$
\begin{aligned}
v_{n} \wedge d \theta_{n} & \equiv 0 \bmod \bar{\alpha}^{2} \\
d v_{n} & \equiv 0 \bmod \alpha^{1}
\end{aligned}
$$

it can be seen that

$$
d \bar{\alpha}^{2} \equiv-\frac{1}{L_{n}} \sec ^{2}\left(\theta_{n}-\theta_{n-1}\right) d \theta_{n-1} \wedge v_{n} \quad \bmod \alpha^{1}, \bar{\alpha}^{2}
$$

Also, since

$$
\alpha^{3} \wedge \pi=L_{n-1} d \theta_{n-1} \wedge v_{n}
$$

a choice of

$$
\bar{\alpha}^{3}=\frac{1}{L_{n} L_{n-1}} \sec ^{2}\left(\theta_{n}-\theta_{n-1}\right) \alpha^{3}
$$


will result in the congruence:

$$
d \bar{\alpha}^{2} \equiv-\bar{\alpha}^{3} \wedge \pi \quad \bmod \alpha^{1}, \bar{\alpha}^{2}
$$

Since the new basis we are defining is merely a scaled version of the original basis, mod-ing out by $\alpha^{i}$ or $\bar{\alpha}^{i}$ is equivalent.

Continuing, we find

$$
\begin{equation*}
d \bar{\alpha}^{3}=d\left(\frac{1}{L_{n}} \sec ^{2}\left(\theta_{n}-\theta_{n-1}\right) d \theta_{n-1}-\frac{1}{L_{n} L_{n-1}} \sec ^{2}\left(\theta_{n}-\theta_{n-1}\right) \tan \left(\theta_{n-1}-\theta_{n-2}\right) v_{n-1}\right) . \tag{31}
\end{equation*}
$$

Referring to Lemma 11 and Equation (29), we see that the following congruences hold,

$$
\begin{aligned}
d \theta_{n} \wedge v_{n} & \equiv 0 \bmod \alpha^{2} \\
d \theta_{n-1} \wedge v_{n} & \equiv 0 \bmod \alpha^{3} \\
d \theta_{n} \wedge d \theta_{n-1} & \equiv 0 \bmod \alpha^{2}, \alpha^{3} \\
d v_{n-1} & \equiv 0 \bmod \alpha^{1}, \alpha^{2}, \alpha^{3}
\end{aligned}
$$

and that using these, Equation (31) can be reduced to

$$
\begin{aligned}
d \bar{\alpha}^{3} & \equiv \frac{1}{L_{n} L_{n-1}} \sec ^{2}\left(\theta_{n}-\theta_{n-1}\right) \sec ^{2}\left(\theta_{n-1}-\theta_{n-2}\right) d \theta_{n-2} \wedge v_{n-1} \bmod \alpha^{1}, \bar{\alpha}^{2}, \bar{\alpha}^{3} \\
& \equiv \frac{1}{L_{n} L_{n-1}} \sec ^{3}\left(\theta_{n}-\theta_{n-1}\right) \sec ^{2}\left(\theta_{n-1}-\theta_{n-2}\right) d \theta_{n-2} \wedge v_{n} \bmod \alpha^{1}, \bar{\alpha}^{2}, \bar{\alpha}^{3}
\end{aligned}
$$

In the second expression we have written $v_{n-1}$ in terms of $v_{n}$. Also, $\alpha^{4} \wedge \pi=$ $L_{n-2} d \theta_{n-2} \wedge v_{n}$, so if we define

$$
\bar{\alpha}^{4}=\frac{-1}{L_{n} L_{n-1} L_{n-2}} \sec ^{3}\left(\theta_{n}-\theta_{n-1}\right) \sec ^{2}\left(\theta_{n-1}-\theta_{n-2}\right) \alpha^{4},
$$

then the congruence

$$
d \bar{\alpha}^{3} \equiv-\bar{\alpha}^{4} \wedge \pi \quad \bmod \alpha^{1}, \bar{\alpha}^{2}, \bar{\alpha}^{3}
$$

results.
In general, we assume that $\bar{\alpha}^{i}$ has been defined as

$$
\bar{\alpha}^{i}=\frac{(-1)^{i-1}}{L_{n} \cdots L_{n-i+2}} \sec ^{i-1}\left(\theta_{n-1}-\theta_{n}\right) \sec ^{i-2}\left(\theta_{n-2}-\theta_{n-1}\right) \cdots \sec ^{2}\left(\theta_{n-i+3}-\theta_{n-i+2}\right) \alpha^{i}
$$

Using the congruences

$$
\begin{aligned}
d \theta_{n-i} \wedge d \theta_{n-i+1} & \equiv 0 \bmod \alpha^{i+2}, \alpha^{i+3} \\
d \theta_{n-i} \wedge v_{n} & \equiv 0 \bmod \alpha^{i+2} \\
d v_{n-i} & \equiv 0 \bmod \alpha^{1}, \ldots, \alpha^{i+2}
\end{aligned}
$$

we can show that

$$
\begin{aligned}
& d \bar{\alpha}^{i} \equiv \frac{(-1)^{i-1}}{L_{n} \cdots L_{n-i+2}} \sec ^{i-1}\left(\theta_{n-1}-\theta_{n}\right) \sec ^{i-2}\left(\theta_{n-2}-\theta_{n-1}\right) \cdots \\
& \sec ^{2}\left(\theta_{n-i+3}-\theta_{n-i+2}\right) \sec ^{2}\left(\theta_{n-i+2}-\theta_{n-i+1}\right) d \theta_{n-i+1} \wedge v_{n-i+2} \\
& \quad \bmod \alpha^{1}, \bar{\alpha}^{2}, \ldots, \bar{\alpha}^{i} \\
& \equiv \frac{(-1)^{i-1}}{L_{n} \cdots L_{n-i+2}} \sec ^{i}\left(\theta_{n-1}-\theta_{n}\right) \sec ^{i-1}\left(\theta_{n-2}-\theta_{n-1}\right) \cdots \\
& \sec ^{3}\left(\theta_{n-i+3}-\theta_{n-i+2}\right) \sec ^{2}\left(\theta_{n-i+2}-\theta_{n-i+1}\right) d \theta_{n-i+1} \wedge v_{n} \\
& \bmod \alpha^{1}, \bar{\alpha}^{2}, \ldots, \bar{\alpha}^{i} \\
& \equiv-\bar{\alpha}^{i+1} \wedge v_{n} \bmod \alpha^{1}, \bar{\alpha}^{2}, \ldots, \bar{\alpha}^{i} .
\end{aligned}
$$

All that remains now is to demonstrate that

$$
d \bar{\alpha}^{n+1} \neq 0 \quad \bmod I
$$

From the above analysis, we know

$$
\begin{gathered}
d \bar{\alpha}^{n+1} \equiv \frac{(-1)^{n}}{L_{n} \cdots L_{1}} \sec ^{n+1}\left(\theta_{n-1}-\theta_{n}\right) \cdots \sec ^{3}\left(\theta_{2}-\theta_{1}\right) \sec ^{2}\left(\theta_{1}-\theta_{0}\right) d \theta_{0} \wedge v_{n} \\
\bmod \alpha^{1}, \ldots, \bar{\alpha}^{n+1}
\end{gathered}
$$

which is nonzero.
Now that we have shown that the one-forms $\alpha^{i}$ do satisfy the Goursat congruences, we can follow the steps of the algorithm of Section 2 to find the coordinate transformation that will result in Goursat normal form. Following Algorithm 1, in step 2 we look for possibly non-unique functions $f_{1}, f_{2}$ which satisfy (10), namely

$$
\begin{array}{rlrl}
d \alpha^{1} \wedge \alpha^{1} \wedge d f_{1} & =0 & \text { and } & \alpha^{1} \wedge d f_{1} \neq 0  \tag{10}\\
\alpha^{1} \wedge d f_{1} \wedge d f_{2}=0 & d f_{1} \wedge d f_{2} \neq 0
\end{array}
$$

Since $\alpha^{1}=\sin \theta_{n} d x_{n}-\cos \theta_{n} d y_{n}$ and $d \alpha^{1}=-\cos \theta_{n} d x_{n} \wedge d \theta_{n}-\sin \theta_{n} \wedge d \theta_{n}$, it follows that $d \alpha^{1} \wedge \alpha^{1}=d x_{n} \wedge d y_{n} \wedge d \theta_{n}$. Thus $f_{1}$ may be chosen to be any function of $x_{n}, y_{n}, \theta_{n}$ exclusively. We now proceed to explain two different solutions of the equations (10):
Transformation 1: Coordinates of the $N^{\text {th }}$ trailer. In a choice motivated by Sørdalen [32] we choose $f_{1}=x_{n}$. Then, the second equation of (10) becomes

$$
\sin \theta_{n} d x_{n} \wedge d y_{n} \wedge d f_{2}=0
$$

with the proviso that $d f_{1} \wedge d f_{2} \neq 0$. A non-unique choice of $f_{2}$ is

$$
f_{2}=y_{n}
$$

For the change of coordinates, we choose

$$
\begin{aligned}
z_{1} & =f_{1}(x)=x_{n} \\
z_{n+3} & =f_{2}(x)=y_{n} .
\end{aligned}
$$

The one form $\alpha^{1}=0$ may be written by dividing through by $\sin \theta_{n}$ as

$$
\begin{aligned}
\alpha^{1} & =d y_{n}+\tan \theta_{n} d x_{n} \\
& =d z_{n+3}-z_{n+2} d z_{1},
\end{aligned}
$$

so that

$$
z_{n+2}=-\tan \theta_{n} .
$$

By the proof of Engel's theorem, we now need to find $a(x), b(x)$ so that:

$$
\begin{aligned}
\alpha^{2} & \equiv a(x) d z_{n+2}+b(x) d z_{1} & & \bmod \alpha^{1} \\
& =-a(x) \sec ^{2}\left(\theta_{n}\right) d \theta_{n}+b(x) d x_{n} & & \bmod \alpha^{1} .
\end{aligned}
$$

But $\alpha^{2}=L_{n} d \theta_{n}-\tan \left(\theta_{n}-\theta_{n-1}\right) v_{n}$. Hence, we have that

$$
a(x)=\frac{-L_{n}}{\sec ^{2} \theta_{n}} \quad b(x)=\frac{-\tan \left(\theta_{n}-\theta_{n-1}\right)}{\cos \theta_{n}},
$$

and we may write

$$
\alpha^{2}=d z_{n+2}-\frac{-b(x)}{a(x)} d z_{1}
$$

Now, we define

$$
z_{n+1}:=-\frac{b(x)}{a(x)}=\frac{\tan \left(\theta_{n}-\theta_{n-1}\right) \cos \theta_{n}}{L_{n}}
$$

The remaining coordinates are found by solving the equations

$$
\alpha^{i}=d z_{n-i+4}-z_{n-i+3} d z_{1} \quad \bmod \alpha^{1}, \ldots, \alpha^{i-1}
$$

for $i \geq 2$. The details are not particularly insightful and are omitted here.
Transformation 2: Coordinates of the origin seen from the last trailer. Yet another choice for $f_{1}$ corresponds to writing the coordinates of the origin as seen from the last trailer. This is reminiscent of a transformation used by Samson [31] in a different context, and is given by

$$
z_{1}:=f_{1}(x)=x_{n} \cos \theta_{n}+y_{n} \sin \theta_{n} .
$$

This has the physical interpretation of being the origin of the reference frame when viewed from a coordinate frame attached to the $n^{\text {th }}$ trailer. It satisfies the first of the equations of (10) simply by virtue of the fact that it is a function of $x_{n}, y_{n}, \theta_{n}$. It may be verified that a choice of $f_{2}$ (non-unique-we got it by guess work!) given by

$$
z_{n+3}:=f_{2}=x_{n} \sin \theta_{n}-y_{n} \cos \theta_{n}-\theta_{n} z_{1}
$$

satisfies

$$
\alpha^{1} \wedge d f_{1} \wedge d f_{2}=0
$$

The remaining coordinates $z_{2}, \ldots, z_{n+2}$ corresponding to this transformation may be obtained from the same procedure as in the previous solution. The details are
tedious. ${ }^{3}$ In the next subsection, we discuss yet another technique for obtaining the coordinates for the Goursat normal form.
3.3. Conversion to Chained Form. In the previous section, we described a method for converting the $N$-trailer exterior differential system into Goursat normal form. Recalling from Section 2 that the dual of Goursat normal form is chained form, we now show how a similar procedure can be used to transform the the nonholonomic control system corresponding to the $N$-trailer system into chained canonical form.

We note that an exterior differential system on $\mathbb{R}^{n}$ of codimension 2 , given by

$$
I=\left\{\alpha^{1}(x), \ldots, \alpha^{n-2}(x)\right\}
$$

is dual to a two-input nonholonomic control system:

$$
\begin{equation*}
\Sigma: \quad \dot{x}=g_{1}(x) u_{1}+g_{2}(x) u_{2}, \tag{32}
\end{equation*}
$$

where the vector fields $g_{j}(x)$ span a 2-dimensional distribution $\Delta$ which is annihilated by the one-forms $\alpha^{i}$ :

$$
\alpha^{i}(x) \cdot g_{j}(x)=0 .
$$

When we transform an exterior differential system into Goursat normal form, we only perform a coordinate transformation $z=f(x)$. There is no input per se to a formal exterior differential system, although we can speak of the two degrees of freedom of the system, given by the distribution $\Delta=I^{\perp}$.

The procedure for transforming a nonholonomic control system such as (32) into chained form requires both a coordinate transformation and state feedback. Although for the most general case, a state feedback is given by

$$
\bar{u}=a(x)+b(x) u
$$

for drift-free nonholonomic systems it is easily seen that $a(x)=0$. (If this were not the case, the state feedback would add a drift term to a drift-free system and could not result in a chained form.) The purpose of the state feedback $\bar{u}=b(x) u$ is therefore to transform the basis of the distribution $\Delta$ into chained form in the new coordinate system:

$$
\begin{align*}
& \bar{g}_{1}(z)=\frac{\partial}{\partial z_{1}}+z_{2} \frac{\partial}{\partial z_{3}}+\cdots+z_{n-1} \frac{\partial}{\partial z_{n}} \\
& \bar{g}_{2}(z)=\frac{\partial}{\partial z_{2}} . \tag{33}
\end{align*}
$$

In this section, we follow through the calculations for transforming the nonholonomic control system of the $N$-trailer problem into chained form. Although some of the details are very similar to what has already been presented in Section 3.2, we want to highlight the distinctions between the exterior differential and the vector field formulations of the system, and we feel that an involved discussion is merited.

[^2]First, we will find a basis for $\Delta=I^{\perp}$ and show that the two vector fields which we choose as the basis have a physical meaning in the multi-trailer robotic system. We then consider the problem of putting the $N$-trailer control system into chained form: given the two vector fields $g_{1}, g_{2}$ and inputs $u_{1}, u_{2}$, with the system defined as

$$
\dot{x}=g_{1}(x) u_{1}+g_{2}(x) u_{2},
$$

find a coordinate transform $z=f(x)$ and an input transformation $\bar{u}=b(x) u$ such that the system

$$
\dot{z}=\bar{g}_{1}(z) \bar{u}_{1}+\bar{g}_{2}(z) \bar{u}_{2}
$$

is in chained form, i.e. $\bar{g}_{1}(z), \bar{g}_{2}(z)$ are of the form (33). We present the two coordinate transformations which were defined in the previous section along with the required input transformations, showing that as expected they result in a chained form system, and we also demonstrate that these coordinate transformations are local diffeomorphisms.

Proposition 13. Consider an $N$-trailer system with $n+1$ rolling constraints $\alpha^{i}=0$,

$$
\begin{aligned}
\alpha^{1}(x) & =\sin \theta_{n} d x_{n}-\cos \theta_{n} d y_{n}=0 \\
\alpha^{n+1-i}(x) & =L_{i+1} d \theta_{i+1}-\tan \left(\theta_{i+1}-\theta_{i}\right) v_{i+1}=0 \quad i=0, \ldots, n-1,
\end{aligned}
$$

where the $v_{i}$ are as specified in (19). A basis for the distribution $\Delta$ which is annihilated by these one-forms $\left\{\alpha^{1}, \ldots, \alpha^{n+1}\right\}$ is given by

$$
g_{1}=\left[\begin{array}{c}
\cos \theta_{n} \\
\sin \theta_{n} \\
\frac{1}{L_{n}} \tan \left(\theta_{n-1}-\theta_{n}\right) \\
\vdots \\
\frac{1}{L_{1}} \prod_{i=2}^{n} \sec \left(\theta_{i-1}-\theta_{i}\right) \tan \left(\theta_{0}-\theta_{1}\right) \\
0
\end{array}\right] \quad g_{2}=\left[\begin{array}{c}
0 \\
0 \\
0 \\
\vdots \\
0 \\
1
\end{array}\right] .
$$

Proof. For the proof, we will derive the constraints $\alpha^{i}$ in a different way than was done in Section 3.1. The set of constraints that we use here comes from the condition that the pair of wheels on the $i^{i t h}$ trailer cannot slide sideways; see Figure 4 for the definition of the variables associated with the $i^{\text {th }}$ trailer. The linear velocity of the $i^{\text {th }}$ body is $v_{i}=\dot{x}_{i} \cos \theta_{i}+\dot{y}_{i} \sin \theta_{i}$ and the velocity of the trailer in the direction perpendicular to the wheels is $v_{i}^{\perp}=\dot{x}_{i} \sin \theta_{i}-\dot{y}_{i} \cos \theta_{i}$. The non-slipping constraint requires that $v_{i}^{\perp}=0$.

As stated in the beginning of Section 3.1, $\left(x_{i}, y_{i}\right)$ can be expressed in terms of $\left(x_{n}, y_{n}, \theta_{n}, \ldots, \theta_{i}\right):$

$$
\begin{aligned}
x_{i} & =x_{i+1}+L_{i+1} \cos \theta_{i+1} \\
y_{i} & =y_{i+1}+L_{i+1} \sin \theta_{i+1}
\end{aligned}
$$

to determine that


Figure 4. The $i^{\text {th }}$ body, showing the velocities. The velocity $v_{i}$ is defined to be in the same direction the wheels are pointing, and so the velocity perpendicular to this direction, $v_{i}^{\perp}$, must be zero if the rolling constriants are to be satisfied.

$$
v_{i}^{\perp}=\dot{x}_{n} \sin \theta_{i}-\dot{y}_{n} \sin \theta_{i}-\sum_{k=i+1}^{n} L_{k} \cos \left(\theta_{i}-\theta_{k}\right) \dot{\theta}_{k}
$$

Once again we will abuse notation and also use $v_{i}^{\perp}$ to refer to the one-form:

$$
v_{i}^{\perp}=\sin \theta_{i} d x_{n}-\sin \theta_{i} d y_{n}-\sum_{k=i+1}^{n} L_{k} \cos \left(\theta_{i}-\theta_{k}\right) d \theta_{k}
$$

Remark 2. The one-forms defined by these velocity constraints

$$
\omega^{i}:=v_{n-i+1}^{\perp}=0, \quad i=1, \ldots, n+1
$$

are also adapted to the derived flag. Indeed, since the one-forms $\alpha^{i}$ as defined in Equation (23) are adapted to the derived flag, and the relations between the $\omega^{i}$ and the $\alpha^{i}$ are "triangular,"

$$
\omega^{i}=\alpha^{i}+\sum_{k=1}^{i-1} c_{k}^{i} \alpha^{k}
$$

for some coefficient functions $c_{k}^{i}$, it follows that the $\omega^{i}$ are adapted to the derived flag as well.

Because $\left\{\omega^{1}, \ldots, \omega^{n+1}\right\}$ form an $n+1$ dimensional co-distribution $\Omega$ on $T^{*} M$, there exists a 2 -dimensional distribution $\Delta$ on $M$ which is annihilated by $\Omega$. A basis for this distribution is given by two linearly independent vector fields $g_{1}, g_{2}$ which satisfy:

$$
\omega^{i}(x) \cdot g_{j}(x)=0 \quad \forall i=0, \ldots, n, j=1,2
$$

Since none of the $\omega^{i}$ have a term $d \theta_{0}$, one of the vector fields in $\Delta$ can be chosen to
be be

$$
g_{2}=\frac{\partial}{\partial \theta_{0}}
$$

## $0^{0}$

It can be verified that choosing the other vector field

$$
g_{1}=\cos \theta_{n} \frac{\partial}{\partial x_{n}}+\sin \theta_{n} \frac{\partial}{\partial y_{n}}+\sum_{k=1}^{n} \frac{1}{L_{k}} \tan \left(\theta_{k-1}-\theta_{k}\right) \prod_{i=k+1}^{n} \sec \left(\theta_{i-1}-\theta_{i}\right) \frac{\partial}{\partial \theta_{k}}
$$

will result in $\omega^{i} \cdot g_{1}=0 \forall i$. In a more familiar notation, these two vector fields are written as

$$
g_{1}=\left[\begin{array}{c}
\cos \theta_{n} \\
\sin \theta_{n} \\
\frac{1}{L_{n}} \tan \left(\theta_{n-1}-\theta_{n}\right) \\
\vdots \\
\frac{1}{L_{1}} \prod_{i=2}^{n} \sec \left(\theta_{i-1}-\theta_{i}\right) \tan \left(\theta_{0}-\theta_{1}\right) \\
0
\end{array}\right] \quad g_{2}=\left[\begin{array}{c}
0 \\
0 \\
0 \\
\vdots \\
0 \\
1
\end{array}\right]
$$

where the coordinates are written in the order $x=\left(x_{n}, y_{n}, \theta_{n}, \ldots, \theta_{0}\right)$.
Although there are many different choices of $g_{1}, g_{2}$ which will span $\Delta$, the two which we have picked are natural in the sense that when the nonholonomic control system is written as:

$$
\dot{x}=g_{1}(x) u_{1}+g_{2}(x) u_{2}
$$

the input functions have the physical meanings: $u_{1}=v_{n}$ is the linear velocity of the $n^{t h}$ trailer, and $u_{2}=\omega$ is the rotational velocity of the lead car. From a practical point of view, we have control only on the velocity $v_{0}$ of the lead car given in terms of $v_{n}$ by

$$
v_{0}=\sec \left(\theta_{0}-\theta_{1}\right) \sec \left(\theta_{1}-\theta_{2}\right) \cdots \sec \left(\theta_{n-1}-\theta_{n}\right) v_{n}
$$

This is merely an input transformation, and will not change any of the properties of the chained form system.

We will now derive the coordinate transformations and changes of input required to put the system into chained form, as was discussed in Section 2.7. Recall that a system in chained canonical form is defined to be

$$
\begin{aligned}
\dot{z}_{1} & =u_{1} \\
\dot{z}_{2} & =u_{2} \\
\dot{z}_{3} & =z_{2} u_{1} \\
& \vdots \\
\dot{z}_{m} & =z_{m-1} u_{1} .
\end{aligned}
$$

We note that the functions $z_{1}(t)$ and $z_{m}(t)$ will completely define all the state variables of a chained-form system, ${ }^{4}$ since the other $m-2$ states and the two inputs

[^3]can be determined from the equations:
\[

$$
\begin{align*}
u_{1} & =\dot{z}_{1} \\
z_{i} & =\dot{z}_{i+1} / u_{1} \quad i=m-1, \ldots, 2  \tag{34}\\
u_{2} & =\dot{z}_{2}
\end{align*}
$$
\]



Consequently, a coordinate transformation into chained form is completely defined by the first and last coordinates of the chain, $z_{1}$ and $z_{m}$, as functions of the original coordinates $x$ along with equation (34). (The fact that such a transform exists follows from our having verified the Goursat congruences for the $\alpha^{i}$ in the previous subsection.) It does need to be checked that the transformation which results from equation (34) is a valid diffeormorphism. In general, there are many possible transformations into chained form; two are presented here. These two are exactly the same as those discussed in the previous subsection in the context of the Goursat normal form.
Transformation 1: Coordinates of the $N^{\text {th }}$ trailer. Originally proposed by Sørdalen [32], and also used in the previous section, is as follows:

$$
\begin{aligned}
z_{1} & =x_{n} \\
z_{n+3} & =y_{n} .
\end{aligned}
$$

The corresponding input transformation is:

$$
\bar{u}_{1}=\dot{z}_{1}=\cos \theta_{n} v_{n}=\cos \left(\theta_{0}-\theta_{1}\right) \cos \left(\theta_{1}-\theta_{2}\right) \cdots \cos \left(\theta_{n-1}-\theta_{n}\right) v_{0}
$$

The other input $\bar{u}_{2}=\dot{z}_{2}$ is a quite complicated function of $x, v_{0}, \omega$ for the general case with $n$ trailers. However, it is easily verified that

$$
\frac{\partial \bar{u}_{2}}{\partial \omega} \neq 0
$$

implying that the input transformation $\bar{u}=b(x) u$ is nonsingular. The remaining coordinates $z=f(x)$ are defined using equation (34); Mathematica code which generates these coordinates symbolically is given in Appendix A.

It can be checked that this coordinate transformation is valid by looking at the Jacobian,

$$
\left[\frac{\partial z}{\partial x}\right]=\left[\begin{array}{cc|ccc}
1 & 0 & 0 & \cdots & 0 \\
0 & 1 & 0 & \cdots & 0 \\
\hline 0 & 0 & * & & 0 \\
\vdots & \vdots & & \ddots & \\
0 & 0 & * & & *
\end{array}\right]
$$

where the coordinates are written in the order: $x=\left(x_{n}, y_{n}, \theta_{n}, \theta_{n-1}, \ldots, \theta_{0}\right), z=$ $\left(z_{1}, z_{n+3}, z_{n+2}, \ldots, z_{2}\right)$ and $*$ represents any nonzero function. The ordering of the $z$ coordinates was chosen to put the Jacobian matrix in a lower-triangular form, thereby highlighting its nonsingularity. That the Jacobian is nonsingular implies that the map $f: x \rightarrow z$ is a local diffeomorphism.


It should be noted that this coordinate transformation is only defined locally. Since its definition requires a division by $u_{1}$, if any of the factors in $u_{1}$ are zero, the transformation is undefined for that particular configuration. For example, if $\theta_{n}=\pi / 2$, corresponding to the last trailer being at right-angles with the coordinate frame, this coordinate transformation is no longer valid. In addition, if the $i^{\text {th }}$ trailer is jack-knifed, that is to say, for some $1 \leq i \leq n, \quad \theta_{i}=\theta_{i-1} \pm \pi / 2$, the coordinate transformation is also singular.
Transformation 2: Coordinates of the origin as seen from the last trailer. Another coordinate transformation which also has some singularities but will allow the trailer to be at any orientation with respect to the coordinate frame, was also detailed in the previous section in the forms version; we define it here as:

$$
\begin{aligned}
z_{1} & =x_{n} \cos \theta_{n}+y_{n} \sin \theta_{n} \\
z_{n+3} & =x_{n} \sin \theta_{n}-y_{n} \cos \theta_{n}-\theta_{n} z_{1} .
\end{aligned}
$$

The input transformation and the rest of the coordinates follow from Equation (34), Mathematica code which generates both the coordinate and input transformations is given in Appendix A. Once again, it can be verified that the input transformation has the form:

$$
\binom{\bar{u}_{1}}{\bar{u}_{2}}=\left[\begin{array}{cc}
b_{1,1}(x) & 0 \\
b_{2,1}(x) & b_{2,2}(x)
\end{array}\right]\binom{v_{0}}{\omega}
$$

with $b_{1,1}$ and $b_{2,2}$ nonzero functions of $x$. This implies that the input transformation is nonsingular.

We can show that this coordinate transformation is nonsingular by looking at its Jacobian:

$$
\left[\frac{\partial z}{\partial x}\right]=\left[\begin{array}{ccc|ccc}
\cos \theta_{n} & \sin \theta_{n} & * & 0 & & 0 \\
\sin \theta_{n} & -\cos \theta_{n} & * & & \ddots & \\
0 & 0 & -1 & 0 & & 0 \\
\hline 0 & 0 & * & * & & 0 \\
\vdots & \vdots & \vdots & & \ddots & \\
0 & 0 & * & * & & *
\end{array}\right]
$$

where the coordinates are written in the order: $x=\left(x_{n}, y_{n}, \theta_{n}, \theta_{n-1}, \ldots, \theta_{0}\right)$ and $z=\left(z_{1}, z_{n+3}, z_{n+2}, \ldots, z_{2}\right)$ and $*$ represents any nonzero function. Again, since the Jacobian is nonsingular, the map $f: x \rightarrow z$ is a local diffeomorphism. The singularities in this transformation also occur when division by $u_{1}$ is undefined. This happens when the expression

$$
L_{n}+\left(y \cos \theta_{n}-x \sin \theta_{n}\right) \tan \left(\theta_{n}-\theta_{n-1}\right)=0,
$$

and also when any of the trailers is jack-knifed.


Figure 5. The front-wheel drive car with $n$ trailers. This model is similar to that of Hilare 2 with an extra axle added to the front of the first body in the chain.
3.4. Generalizations. Thus far, we have concentrated our attention on the example of the Hilare mobile robot pulling a chain of trailers. In this section we demonstrate that this model is equivalent (under a coordinate transformation and state feedback) not only to the more familiar system of a front-wheel drive car pulling trailers, but also to the luggage trains commonly found in airports.

The model of the front-wheel drive car is shown in Figure 5. In comparison with the Hilare model, we have added another axle to the front body of the chain, and a variable $\phi$ representing the angle of the front wheels with respect to the car. The length of the wheelbase of the lead car is defined to be $L_{0}$.

The equivalence between the two models is most easily seen by looking at the form constraints. Each constraint corresponds to one axle rolling without slipping. Hilare with $n$ trailers has $n+1$ axles; the car with $n$ trailers has $n+2$ axles, and its Pfaffian system is therefore equivalent to that of Hilare pulling $n+1$ trailers.

Of course, the states and inputs that we define for the car system are slightly different. By convention, we define the angle of the front axle relative to the car instead of relative to the coordinate frame. This angle $\phi$ is merely $\theta_{0}-\theta_{1}$ on the Hilare system. The velocity input is the same, assumed to be the linear velocity of the first body (we can define it at either the front or rear axle depending on whether our car is front-wheel drive or rear-wheel drive), but the rotational input is usually taken as $\omega^{\prime}=\dot{\phi}$ the steering wheel velocity. Since in the Hilare case, we can control the velocity of the first body $\omega=\dot{\theta}_{0}$, state feedback can be used to reconcile these differences. As mentioned in the proof of Proposition 13, there are many choices of vector fields orthogonal to a given Pfaffian system with each choice having a different physical meaning.

The luggage carts used at most airports are also equivalent to the Hilare model. Each trailer on the luggage cart train has two sets of wheels; the front axle can spin freely about its center but the back axle is constrained to be aligned with the trailer (see Figure 6). Here again we have defined the angles of the front wheels with respect to each trailer, but looking at the form constraints it is easily seen that the cab with $n$ luggage trailers is equivalent to a front-wheel drive car with $2 n$ one-axle trailers. Again, a coordinate transformation is needed, since in the model of the luggage carts we define the angle of the front wheels of the trailers relative to the trailer instead of relative to the coordinate frame.


Figure 6. A car pulling $n$ luggage carts. Each trailer has two axles; the front axle is free to spin about its midpoint but the rear axle is constrained to be aligned with the body of the trailer.

## 4. Steering Chained Form Systems

Now that we have seen how to transform an N -trailer system into chained form, we examine various methods for steering chained form systems:

$$
\begin{align*}
\dot{z}_{1} & =u_{1} \\
\dot{z}_{2} & =u_{2} \\
\dot{z}_{3} & =z_{2} u_{1}  \tag{35}\\
& \vdots \\
\dot{z}_{m} & =z_{m-1} u_{1} .
\end{align*}
$$

We assume an $m$-state system, and note that Hilare with $n$ trailers has $n+3$ states, a car with $n$ trailers has $n+4$ states, and a car with $n$ luggage trailers has $2 n+4$ states.

The problem that we address in this section is: Given a system in chained form with an initial state $z^{0}$ and a goal state $z^{j}$, find some control inputs $u_{1}(t), u_{2}(t)$ which will steer the system from $z^{0}$ to $z^{j}$ after some time $T$. The application of these results to the problem of steering the mobile robot with multiple trailers is covered in the next section.

We present three methods to steer the chained form system:
(1) Sinusoidal inputs
(2) Piecewise constant inputs
(3) Polynomial inputs
4.1. Sinusoidal inputs. The first steering method that we consider uses sinusoidal inputs. Steering chained form systems with sinusoids was originally proposed by us in [29]. The method that we have developed here is different from the original algorithm in that it steers all the states in one step, instead of one state at a time.

Given an $m$-state chained form system, it is easily seen that the first two states, $z_{1}$ and $z_{2}$, can be steered from their initial to their final positions using constant inputs over any time period $T$. Of course, the states $z_{3}, \ldots, z_{m}$ will drift as a consequence of this.

By direct integration, it may be verified that a combination of out of phase sinusoids applied to the inputs,

$$
u_{1}(t)=\alpha \sin \omega t \quad u_{2}(t)=\beta \cos \omega t
$$


over one period $T=2 \pi / \omega$, will cause a motion in the $z_{3}$ variable as follows:

$$
\begin{aligned}
& z_{1}(T)=z_{1}(0) \\
& z_{2}(T)=z_{2}(0) \\
& z_{3}(T)=z_{3}(0)+\frac{\alpha \beta}{2 \omega} .
\end{aligned}
$$

The states $z_{4}, \ldots, z_{m}$ will drift in some fashion. Further, using inputs with $u_{2}$ having $k$ times the frequency of $u_{1}$, namely:

$$
u_{1}(t)=\alpha \sin \omega t \quad u_{2}(t)=\beta \cos k \omega t
$$

applied over one period $T=2 \pi / \omega$, will result (as may be verified directly by integration) to be

$$
\begin{aligned}
z_{1}(T) & =z_{1}(0) \\
& \vdots \\
z_{k+1}(T) & =z_{k+1}(0) \\
z_{k+2}(T) & =z_{k+2}(0)+\frac{\alpha^{k} \beta}{k!(2 \omega)^{k}}
\end{aligned}
$$

The intuition behind this steering scheme lies in the different levels of Lie brackets. If we consider the input vector fields $g_{1}, g_{2}$, we note that $\left[g_{1}, g_{2}\right]=\left[\begin{array}{llll}0 & 1 & 1 & \cdots\end{array}\right]^{T}$, precisely in the $z_{3}$ direction. Motion in this first level Lie bracket is generated by cycling between the two input vector fields in a continuous manner described by the out of phase sinusoids. To get motion in the second level Lie bracket, $\left[g_{1},\left[g_{1}, g_{2}\right]\right]=$ [ $00010 \cdots 0]^{T}$ or equivalently the $z_{4}$ direction, the input $u_{2}$ completes two cycles for one cycle on $u_{1}$. More generally, motion in the $\mathrm{ad}_{g_{1}}^{k} g_{2}=[0 \cdots 1 \cdots 0]^{T}$ or the $z_{k+2}$ direction is achieved by using $k$ times the frequency of $u_{1}$ on $u_{2}$.

The Murray and Sastry steering algorithm is step-by-step: It first steers $z_{1}, z_{2}$ to their final position using constant inputs, disregarding the other states. Then it steers $z_{3}$ to its desired final position using sinusoids, $z_{1}, z_{2}$ will return to their final values. Now $z_{4}$ can be steered, and similarly on down the chain, until all states are at their final positions. This is a simple algorithm that is easy to implement, but can be time-consuming when there are many states to be steered.

We propose instead an "all-at-once" sinusoids method, combining all the frequencies on $u_{2}$ together in one step,

$$
\begin{align*}
& u_{1}=a_{0}+a_{1} \sin \omega t \\
& u_{2}=b_{0}+b_{1} \cos \omega t+b_{2} \cos 2 \omega t+\cdots+b_{m-2} \cos (m-2) \omega t . \tag{36}
\end{align*}
$$

It is no longer as simple to choose appropriate values for the parameters ( $a_{0}, a_{1}, b_{0}$, $\ldots, b_{m-2}$ ) because of the drift that we were able to ignore when we considered
each state individually. However, it is still possible to integrate the chained form equations sequentially, finding $z_{1}(t), z_{2}(t), z_{3}(t), \ldots, z_{m}(t)$ which result from the inputs (36) above. The state $z(t)$ is a function of the initial condition $z^{0}$ as well as the input parameters $a_{0}, a_{1}, b_{0}, \ldots, b_{m-2}$. If we evaluate $z(T)$, with $T=2 \pi / \omega$, all the sinusoidal functions will evaluate to either 0 or 1 . By setting $z(T)=z^{f}$ we get a set of $m$ polynomial equations in the ( $m+1$ ) input parameters ( $a_{0}, a_{1}, b_{0}, \ldots, b_{m-2}$ ). The following proposition guarantees the existence of solutions to these equations at least locally around $z^{0}$.

Proposition 14. Consider an m-state chained form system with initial and final states $z^{0}, z^{f}$. If $\left|z^{0}-z^{f}\right|<\delta$ small, then there exist input parameters $\left(a_{0}, a_{1}, b_{0}, \ldots, b_{m-2}\right)$ such that the inputs

$$
\begin{aligned}
& u_{1}=a_{0}+a_{1} \sin \omega t \\
& u_{2}=b_{0}+b_{1} \cos \omega t+b_{2} \cos 2 \omega t+\cdots+b_{m-2} \cos (m-2) \omega t
\end{aligned}
$$

will steer the system from $z^{0}$ to $z^{f}$ in time $T=2 \pi / \omega$.
Proof. Consider the map

$$
\phi_{z^{0}}: \mathbb{R}^{m} \rightarrow \mathbb{R}^{m}
$$

which takes values in the parameter space $\left(a_{0}, b_{0}, \ldots, b_{m}\right)$ and maps them to values in the state space $\left(z_{1}^{f}, \ldots, z_{m}^{f}\right)$. We define $\phi_{z} 0\left(a_{0}, b_{0}, \ldots, b_{m}\right)$ to be the value of $z(T)$ when the chained form system (35) is integrated starting at the initial condition $z^{0}$ and applying the inputs (36) over the time period $[0, T]$. We choose $a_{1} \neq 0$. We will show that $\phi_{z^{\circ}}$ is a local diffeomorphism by looking at its directional derivatives, and demonstrating that the Jacobian of $\phi_{20}$ is nonsingular.

Let $\left\{e_{i}\right\}_{i=1}^{m}$ be the standard basis for $\mathbb{R}^{m}$ and let $\epsilon$ be small. Set $a_{1} \neq 0$. Now consider the input parameterized by $\epsilon e_{1}$,

$$
u_{1}=\epsilon+a_{1} \sin \omega t \quad u_{2}=0 .
$$

Integrating the chained form equations and evaluating it at time $T$ will give

$$
\phi_{z^{0}}\left(\epsilon e_{1}\right)=z^{0}+\left[\begin{array}{lll}
\epsilon T & 0 & o(\epsilon) \cdots o(\epsilon)
\end{array}\right]^{T}
$$

where $o(\epsilon)$ represents terms that are of linear and higher order in $\epsilon$.
Now consider the input parameterized by $\epsilon e_{2}$

$$
u_{1}=a_{1} \sin \omega t \quad u_{2}=\epsilon
$$

We integrate and evaluate at $T$ as before,

$$
\phi_{z^{0}}\left(\epsilon e_{2}\right)=z^{0}+\left[\begin{array}{lll}
0 & \epsilon T & o(\epsilon) \cdots o(\epsilon)
\end{array}\right]^{T} .
$$

In this case it may be verified that $o(\epsilon)$ terms are linear in $\epsilon$. In general, for an input parameterized by $\epsilon e_{k}$,

$$
u_{1}=a_{1} \sin \omega t \quad u_{2}=\epsilon \cos (k-2) \omega t
$$




Figure 7. The inputs and state trajectories for a six-state, chained form system, steering from ( $-10,-7,-2,2,4,8$ ) to the origin. The input $u_{1}$ is sinusoidal of one period; $u_{2}$ is a sum of sinusoids, of which the highest frequency is $4 \omega$.
the directional derivative of $\phi$ in the $e_{k}$ direction is given by:

$$
\phi_{z^{0}}\left(\epsilon e_{k}\right)=z^{0}+\left[\begin{array}{lll}
0 & \cdots & p(\epsilon)
\end{array} o(\epsilon) \cdots o(\epsilon)\right]^{T}
$$

where

$$
p(\epsilon)=\frac{a_{1}^{k-2} \epsilon}{(k-2)!(2 \omega)^{k-2}} .
$$

These $m$ directional derivatives are seen to be linearly independent; implying that the Jacobian of $\phi_{z^{\circ}}$ is nonsingular, and that $\phi_{z^{\circ}}$ is a local diffeomorphism.

Remark 3. We have dealt with the overparameterization of the input ( $m+1$ parameters: $a_{0}, a_{1}, b_{0}, \ldots, b_{m-2}$ and $m$ states) by initially choosing a value for $a_{1}$ and then solving the $m$ equations for the remaining $m$ input parameters.

We note here that by choosing a fixed value for $a_{1}$, we are requiring $u_{1}$ to go through one period. Since $u_{1}$ roughly corresponds to the driving input in a mobile robot system, paths planned using the sinusoidal method generally have one backup or speed reversal, corresponding to the zero-crossing of $u_{1}$. Parallel-parking type maneuvers seem particularly well-suited to sinusoidal trajectories.

Remark 4. Appendix A contains Mathematica code which symbolically integrates the chained form system and solves for the input parameters $a_{0}, b_{0}, \ldots, b_{m-2}$ in terms of $a_{1}, \omega$, and the initial and final states $z^{0}, z^{f}$.

A sample of the input functions and state trajectories for a sinusoidal steering problem is shown in Figure 7. There are six states, in chained form, steering from an initial position of $\left(z_{1}, z_{2}, z_{3}, z_{4}, z_{5}, z_{6}\right)=(-10,-7,-2,2,4,8)$ to the origin. The parameters were chosen to be $T=10$ seconds and $a_{1}=\frac{6 \pi}{T}$.
4.2. Piecewise Constant Inputs. The second method we investigate for steering chained form systems uses piecewise constant inputs. This method was originally proposed by Monaco and Normand-Cyrot [26], and was inspired by multirate digital control. It is most easily understood in the context of nonholonomic motion planning simply as piecewise constant inputs.

Consider holding the inputs $u_{1}$ and $u_{2}$ constant over some small time period $[0, \bar{\delta})$,

$$
\begin{aligned}
& u_{1}(\tau)=u_{1,1} \\
& u_{2}(\tau)=u_{2,1} .
\end{aligned} \quad \tau \in[0, \bar{\delta})
$$

The chained form state equations can then be integrated, and evaluated at time $\bar{\delta}$ to yield

$$
\begin{align*}
z_{1}(\bar{\delta}) & =z_{1}(0)+u_{1,1} \bar{\delta} \\
z_{2}(\bar{\delta}) & =z_{2}(0)+u_{2,1} \bar{\delta} \\
z_{3}(\bar{\delta}) & =z_{3}(0)+z_{2}(0) u_{1,1} \bar{\delta}+u_{1,1} u_{2,1} \frac{\bar{\delta}^{2}}{2} \\
& \vdots \\
z_{m}(\bar{\delta})= & z_{m}(0)+z_{m-1}(0) u_{1,1} \bar{\delta}+\cdots+u_{2,1} u_{1,1}^{m-2} \frac{\bar{\delta}^{m-1}}{(m-1)!} \tag{37}
\end{align*}
$$

We can now consider another pair of constant inputs on the time interval $[\bar{\delta}, 2 \bar{\delta})$,

$$
\begin{aligned}
& u_{1}(\tau)=u_{1,2} \\
& u_{2}(\tau)=u_{2,2} .
\end{aligned} \quad \tau \in[\bar{\delta}, 2 \bar{\delta})
$$

Integration of the state equations gives us $z(2 \bar{\delta})$ as a function of $z(\bar{\delta}), u_{1,2}, u_{2,2}$. Using $z(\bar{\delta})$ from equation (37), we get an expression for $z(2 \bar{\delta})$ in terms of $z(0), u_{1,1}, u_{1,2}$, $u_{2,1}, u_{2,2}$. This procedure of piecewise integration and substitution can be repeated as many times as necessary.

For path planning, we choose to keep $u_{1}$ at a constant value over the entire trajectory. We therefore iterate the equations (37) $m-1$ times so as to have exactly $m$ parameters for which to solve: $u_{1}, u_{2,1}, \ldots, u_{2, m-1}$. The total time needed for steering is $\delta=(m-1) \bar{\delta}$. Although $\delta$ can be chosen arbitrarily, a smaller time $\delta$ will result in larger inputs $u$ to achieve the same path.

The $m$ equations which result from setting $z(0)=z^{0}$ and $z(\delta)=z^{f}$ are polynomial (of order $m-2$ ) in $u_{1}$ but are linear in $u_{2,1}, \ldots, u_{2, m-1}$. Since $u_{1}$ is easily determined from

$$
u_{1}=\frac{z_{1}^{f}-z_{1}^{0}}{\delta}
$$

the remaining $m-1$ linear equations can be solved for $u_{2}$ quite easily. This is one of the reasons that we propose keeping $u_{1}$ constant over the entire trajectory; if $u_{1}$ varied, we would need to solve high-order polynomial equations in the $u_{1, k}$ parameters.


Figure 8. Sample inputs and state trajectories for steering a sixstate chained form system with Piecewise Constant inputs. The initial position is ( $-10,-7,-2,2,4,8$ ) and the goal point is the origin.

Remark 5. Appendix A contains Mathematica code which will symbolically integrate the chained form equations, and solve for the input values $u_{1}, u_{2, k}$ in terms of the initial and final states $z^{0}, z^{f}$. The code is written for a six-state system, but the generalization to any number of states is straightforward.

It should be noted that if $z_{1}^{f}=z_{1}^{0}$, or the initial and final states agree in the first coordinate, this method as stated so far will fail to yield a solution. From looking at the chained form equations, it is obvious that if $u_{1}=0$, only the second state $z_{2}$ can move; all other states must remain stationary. In practice, this case is dealt with by planning two paths, the first of which takes the initial condition to an intermediate state, the second of which joins the intermediate state with the goal position. The concatenation of these two paths is a valid trajectory between the start and goal. Our algorithm chooses the intermediate point $z^{m}$ halfway between the initial and final points in all coordinates except the first, which we choose to be offset from the starting position by a constant amount,

$$
\begin{aligned}
& z_{k}^{m}=\left(z^{f}-z_{k}^{0}\right) / 2, \quad k=2, \ldots, m \\
& z_{1}^{m}=z_{1}^{0}+\text { const. }
\end{aligned}
$$

The constant offset can be adjusted to fit the situation.
The procedure detailed in the previous paragraph is used when a parallel-parking trajectory is desired for the mobile robot with trailers, since the $z^{1}$ direction in chained form corresponds to "sideways" in the original coordinates. We have found it practical to choose the constant offset at approximately twice the length of the entire robot and trailer system. A smaller offset will result in tighter turns and more lateral motion. If there are obstacles in the field, this constant offset gives a parameter that can be adjusted in an effort to avoid collisions.

Another reason for choosing $u_{1}$ to be constant over the entire trajectory is that in the mobile robot and trailer system, this input is roughly equivalent to the driving velocity. Because of the coordinate transformation that maps $u_{1}$ to the actual

velocity $v_{0}$, the actual velocity of the robot will not be constant, but in most cases it will not cross zero and change sign. This means that the robot will not have to execute backing-up maneuvers to achieve its final goal position.

The main drawback of the piecewise constant inputs is the discontinuity of $u_{2}$. The models used in this paper are purely kinematic using as inputs the driving and steering velocities. In a real robot system, the inputs are not velocities but accelerations, or torques. When a path satisfying the velocity constraints is found, the input velocities need to be differentiated to find the corresponding accelerations. Of their very nature, the piecewise constant trajectories are not differentiable at the switching points.
4.3. Polynomial inputs. Yet another possibility for steering systems in chained form is to use polynomial inputs:

$$
\begin{aligned}
& u_{1}=1 \\
& u_{2}=c_{0}+c_{1} t+\cdots+c_{m-2} t^{m-2}
\end{aligned}
$$

This approach has the advantage of a constant input on $u_{1}$ with the added advantage of the differentiability of $u_{2}$.

The time needed to steer the system from $z^{0}$ to $z^{f}$ is determined by the change desired in the first coordinate,

$$
T=z_{1}^{f}-z_{1}^{0} .
$$

Once $T$ has been found, the state equations (35) can be integrated using the initial condition $z(0)=z^{0}$,

$$
\begin{aligned}
z_{1}(t) & =z_{1}(0)+t \\
z_{2}(t) & =z_{2}(0)+c_{0} t+\frac{c_{1} t^{2}}{2}+\cdots+\frac{c_{m-2} t^{m-1}}{m-1} \\
& \vdots \\
z_{i}(t) & =z_{i}(0)+\sum_{k=0}^{m-2} \frac{k!c_{k} t^{i+k-1}}{(i+k-1)!}+\sum_{k=2}^{i-1} \frac{t^{i-k}}{(i-k)!} z_{k}(0) \\
& \vdots
\end{aligned}
$$

Evaluating the foregoing at time $T$ and setting $z(T)=z^{f}$ yields a total of $m-1$ equations affine in the $m-1$ variables $c_{0}, \ldots, c_{m-2}$,

$$
M(T)\left[\begin{array}{c}
c_{0} \\
c_{1} \\
\vdots \\
c_{m-2}
\end{array}\right]+f(z(0), T)=\left[\begin{array}{c}
z_{2}^{f} \\
z_{3}^{f} \\
\vdots \\
z_{m}^{f}
\end{array}\right]
$$



Figure 9. Sample trajectories and input traces for steering with Polynomial inputs. The initial position is ( $-10,-7,-2,2,4,8$ ) and the goal point is the origin.
where the matrix entries $M_{i, j}(T)$ have the form:

$$
M_{i, j}=\frac{(j-1)!T^{i+j-1}}{(i+j-1)!}
$$

It may be shown that this matrix is nonsingular for $T \neq 0$.
Note that if $z_{1}^{f}-z_{1}^{0}<0$, then we get a solution which gives a negative time period. This situation is easily remedied by choosing $u_{1}=-1$ (see Appendix A for the Mathematica code which solves this problem).

As in the case of steering with piecewise constant inputs, this method will yield no solution when $z_{1}^{f}-z_{1}^{0}=0$. We follow the same procedure outlined in Section 4.2 to deal with this case.
4.4. Other choices. Because of the simple form of the chained form system, many different classes of input functions other than the three described above could be used to steer systems in this form. The chief requirement is that there should be at least as many parameters in the input functions as there are states. For multi-trailer systems, a desirable characteristic of the input functions is that $u_{1}$ have few or no zero-crossings since these will correspond to fewer backups. In fact, the number of backups needed to complete a manoeuvers may be taken as a measure of complexity of an input class.

## 5. Simulations and Observations

We now have an extensive toolbox from which to choose for steering an $N$-trailer system. With two different coordinate transformations into chained form, and at least three different methods for steering the system once it is in chained form, we can try to pick the best combination of coordinate transformation and input type for each start and goal point. There is as yet no formal way to define when one path is "better" than another, but as we mentioned earlier, we tend to think of desirable
paths as those that have few backups and do not stray too far from the vicinity of the start and goal points.

One of the things that must be considered is coordinate singularities. Although we have shown that all three methods proposed here will find a path between any start and goal points in the chained form coordinates, there is no guarantee that this path, when transformed back into the actual coordinates, will avoid the transformation singularities. This must be checked for each desired path. If a singularity does result, another steering method might yield a valid path, or perhaps an intermediate point will need to be chosen, and the path planned in two or more steps.

In Figures 10 and 11, we show two different paths for a front-wheel drive car with two trailers. We have chosen the wheelbase of the car to be $L_{1}=0.5$ units, and each trailer to have a length of $L_{2}=L_{3}=2$ units. Each path was generated using techniques described in this paper: first, transforming the start and goal points into the chained form coordinates; second, steering the chained form system using one of the methods from Section 4; and finally, transforming the trajectory back into the original coordinates.

The trajectory shown in Figure 10 represents the truck backing into a loading dock. The initial condition is $\left(x_{3}, y_{3}, \theta_{3}, \theta_{2}, \theta_{1}, \theta_{0}\right)=(10,10,0,0,0,0)$ and the final position is $\left(0,0, \frac{\pi}{2}, \frac{\pi}{2}, \frac{\pi}{2}, \frac{\pi}{2}\right)$. Coordinate transformation 2 is used, since the first coordinate transformation is singular at the goal position. In the figure, we have presented the trajectory of the front of the car $\left(x_{0}, y_{0}\right)$ instead of the back of the second trailer ( $x_{3}, y_{3}$ ) to amplify the difference between the two steering methods; the trajectories of the second trailer are virtually identical.

In Figure 11 we again have chosen to present the path taken by the front car. Here we have used two different coordinate transformations with the same steering method. The trajectories in the chained form coordinates are identical; however, a difference can be seen in the physical coordinates. Once again, the trajectory traced by the rear of the second trailer is very similar in both cases. Some scenes from a movie animation of this trajectory are shown in Figure 12; in the movie we present the coordinates derived from transformation 1.

With the sinusoidal steering method, there is one parameter that can be adjusted independently of the start and goal positions; this is the magnitude of the sinusoid on the first input, or $a_{1}$ in the terminology of Section 4.1. In constructing this movie, we examined several different values of $a_{1}$; a larger value of $a_{1}$ will correspond to the car driving out farther before it starts backing into the space. We were able to choose a value for this parameter so that the car and trailer system did not hit any of the obstacles along its path.

## 6. Summary and Future Work

In this paper we applied the machinery of exterior differential systems to the $N$ trailer problem. We showed that the multi-trailer system could be put into Goursat normal form, and that this is the dual to chained form. We solved the motion planning problem for the mobile robot pulling $n$ trailers by converting the kinematic equations into chained form, and steering the chained form system from and initial


Figure 10. Backing a car with two trailers into a loading dock. We show here trajectories found by two different steering methods for the same initial and final conditions. The solid line corresponds to the piecewise constant inputs and the dashed line to the polynomial inputs. The $x, y$ trace of the front of the car is shown, since the trajectory of the rear trailer is virtually identical in the two cases. Both trajectories use the second coordinate transformation. The input $v_{0}$ is the dotted line in both graphs. Clips from a movie simulation of this trajectory can be seen in Figure 13 and the movie can be viewed on the left hand pages of this paper.


Figure 11. Parallel-parking a car with two trailers using sinusoids. the trace of the front car is shown for two different choices of coordinates: Transformations 1 (solid line) and 2 (dashed line). We also see how the steering input differs on with the two transformations, although for this path, the driving input $v_{0}$ (dotted line) is similar in both cases.


Figure 12. Scenes from a movie animation, showing the front-wheel drive car with two trailers (a six-state system) parallel-parking in the presence of obstacles. Sinusoidal inputs were used for steering, and the magnitude of the periodic part of the driving input ( $a_{1}$ in the terminology of Section 4) was adjusted so that the obstacles were avoided. The first coordinate transformation was used. The entire movie animation can be seen on the right-hand pages of this paper.


Figure 13. These are scenes from a movie animation, showing the front-wheel drive car with two trailers backing into a loading dock. Piecewise constant inputs were used to steer the chained form system. The entire movie simulation can be viewed on the left-hand pages of this paper.
to a final position, then converting the trajectory back into the original coordinates. Three different methods for steering chained form systems were proposed.

The work done in this paper has several natural avenues of continuation:
(1) The generation of trajectories for the $N$-trailer system in an environment cluttered with obstacles. This line of work has been started in [25] where the authors considered the motion of a single Hilare-like robot in a cluttered environment. Another approach for a Hilare-like robot was defined in [23]. Other methods for obstacle avoidance which use optimization based approaches may be found in $[8,9]$.
(2) The stabilization of open loop trajectories. The trajectories generated by our method need to be stabilized, perhaps using a technique such as that outlined in [36]. There has also been considerable interest in stabilizing nonholonomic systems not to trajectories but to points. Although from Brockett's necessary condition [3] it follows that such stabilizing control laws cannot be both $C^{0}$ and time-invariant, methods using either discontinuous or time-varying feedback have been suggested. One approach to stabilizing chained form nonholonomic systems is given in [34].
(3) Generalized Goursat type canonical forms for exterior differential systems for higher codimension systems are discussed in [12,27] and were useful in transforming to chained form a firetruck system. The firetruck has three inputs: driving and steering in the front and another steering wheel at the tiller [5] (multi-input chained form systems are also discussed in [29]). This work is as yet far from complete since there is a very large number of different possibilities for the normal form in this instance.
(4) There are several examples of nonholonomic systems whose constraints fail to meet the conditions of the Goursat normal form, for example, the system modeling a circular finger tip rolling on a planar face [29]. The problem of steering such systems remains an open one.

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## Appendix A. Mathematica Code

```
(* Strailer.m D. Tilbury November 30,1992
* Defining the coordinate transformation for the system of Hilare
* with three trailers (equivalent to a car with 2 trailers).
*)
(* Definitions of the gradient, Jacobian, and the state vector *)
grad[x.,vect.] := Table[D[x,vect[[jj]]],jj,Length[vect]];
Jac[x,, vect]]:= Table[D[x[[i]],vect[[j]]],i,Length[x],j,Length[vect]];
q = {x,y,th3,th2,th1,th0};
(* The input vector fields g-1 and g_2 *)
g1 ={Cos[th3], Sin[th3], Tan[th2-th3]/L3, Tan[th1-th2] Sec[th2-th3]/L2,
Tan[th0-th1] Sec[th1-th2] Sec[th2-th3]/L1 , 0 };
g2 = {0, 0, 0, 0, 0, 1 };
(* The derivative of the state: the two inputs here are vn, the linear
* velocity of the last body, and omega, the rotational velocity of the
* first body (Hilare)
*)
dq = g1 vn + g2 omega;
(* First define z1 and z6, the first and last chained form coords. *)
(* Coordinate Transform 1 *)
z1 = x;
z6 = y;
(* Coordinate Transform 2
z1 = x Cos[th3] + y Sin[th3];
z6 = x Sin[th3] - y Cos[th3] - th3 z1; *)
(* u1 is then the derivative of z1, and we coax Mathematica to apply
* a useful trig identity *)
u1 = grad[z1,q].dq /. a_ Sin[x] [ 2 + a- Cos[x] ] 2 -> a;
(* Define z5, z4, z3, z2 from the derivatives of z6, z5, z4, z3 *)
z5 = Together[Expand[grad[z6,q].dq/u1]];
z4 = Factor[grad[z5,q].dq/u1];
z3 = Together[grad[z4,q].dq/u1];
z2 = Together[grad[z3,q].dq/u1];
(* then find u2 = dz2 *)
u2 = grad[z2,q].dq;
```



(* The coefficients a0, b0, and b1 can be solved for individually.

* Due to interference, the coefficients b2, b3, and b4 must be
* found together. All of these coefficients will be in terms of
* w and a 1 , both of which are free.
*)
Solve[x1[T] ==xf1,a0] >>> ss_inputs.m
Solve[x2[T]==xf2,b0] >>> ss_inputs.m
Solve[x3[T]==xf3,b1] >>> ss_inputs.m
Solve $[\{x 4[T]==x f 4, x 5[T]==x f 5, x 6[T]==x f 6\},\{b 2, b 3, b 4\}] \ggg$ ss_inputs.m

```
(* Piecewise.m D. Tilbury November 17, 1992
* Here we take a 6-state system in single-chained, 2-input form
* and steer it with Piecevise constant controls over 5 steps,
* d = deltabar = delta/5
*)
(* First calculate the states after one step, using constant controls
*)
x1[i]] := x1[i-1] + d u1[i];
x2[i_] := x2[i-1] + d u2[i];
x3[i_] := x3[i-1] + d x2[i-1] u1[i] + d^2 u2[i] u1[i]/2;
x4[i] := x4[i-1] + d x3[i-1] u1[i] + d^2 x2[i-1] u1[i] 2/2 +
d^3 u1[i]^2 u2[i]/3!;
x5[i] := x5[i-1] + d x4[i-1] u1[i] + d^2 x3[i-1] u1[i] 2/2 +
d^3 x2[i-1] u1[i]`3/3! + d^4 u1[i]^3 u2[i]/4!;
x6[i] := x6[i-1] + d x5[i-1] u1[i] + d`2 x4[i-1] u1[i] 2/2 +
d^3 x3[i-1] u1[i]-3/3! + d^4 x2[i-1] u1[i]^4/4! +
d^5 u1[i]^4 u2[i]/5!;
(* Then ve iterate for 5 steps, *)
x1T = Expand[x1[5]];
x2T = Expand[x2[5]];
x3T = Expand[x3[5]];
x4T = Expand[x4[5]];
x5T = Expand[x5[5]];
x6T = Expand[x6[5]];
(* We define ul to be constant over all 5 time periods *)
u1[2] := u1[1]; u1[3] := u1[1]; u1[4] := u1[1]; u1[5] := u1[1];
(* And nor we solve for the input magnitudes *)
Solve[{x1T == xf1},{u1[1]}] >>> pc_inputs.m
(* We know that xiT are linear in the inputs u2[i], namely
* final = Matrix.input2 + constant
* so we set goal = final and solve for the magnitudes of input2:
* solution = Inverse[Matrix]. (goal - constant)
*)
input2 = {u2[1], u2[2], u2[3], u2[4], u2[5]};
final = { x2T, x3T, x4T, x5T, x6T };
goal = { xf2, xf3, xf4, xf5, xf6 };
Matrix = Jac[final,input2];
constant = final - Matrix.input2;
solution = (Inverse[Matrix].(goal - constant)) >>> pc_inputs.m
```



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[^1]:    ${ }^{1}$ The Hilare family of mobile robots resides at LAAS in Toulouse, see for example [7, 13].
    ${ }^{2}$ The intuition for this comes from the oft repeated dictum: "when backing up a car with a trailer, keep your eye on the hind part of the trailer". Of course, the generalization to this dictum is: when driving a car with $n$ trailers keep your eye on the endpoint of the $n^{\text {th }}$ trailer

[^2]:    ${ }^{3}$ Readers interested in the details of the transformation may obtain it from the first author by email or regular mail.

[^3]:    ${ }^{4}$ As this paper was being finished it was pointed out to the authors that this situation is referred to by Fliess et al. as flat outputs [11].

